Role of body stiffness in undulatory swimming: Insights from robotic and computational models

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The role of body stiffness in undulatory swimming: Insights from robotic and computational models

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Abstract

In an effort to understand the locomotion dynamics of a simple vertebrate, the lamprey, both physical and computational models have been developed. A key feature of these models is the ability to vary the passive stiffness of portions of the swimmer, focusing on highly flexible models similar in material properties to lampreys and other anguilliform fishes. The physical model is a robotic lamprey-like swimmer that is actuated along most of its length but has passively flexible tails of different stiffnesses. The computational model is a two-dimensional model that captures fluid-structure interactions using an immersed boundary framework. This simulated lamprey is passively flexible throughout its length, and is also actuated along most of its length by the activation of muscle forces. Although the three-dimensional robot and the two-dimensional computational swimmer are such different constructs, we demonstrate that the wake structures generated by these models share many features and examine how flexibility affects these features. Both models produce wakes with two or more same-sign vortices shed each time the tail changes direction (a '2P' or higher-order wake). In general, wakes become less coherent as tail flexibility increases. We examine the pressure distribution near the tail tip and the timing of vortex formation in both cases and find good agreement. Because we include flexibility, we are able to estimate resonant frequencies for several of the robotic and computational swimmers. We find that actuation at the resonant frequency dramatically increases the distance traveled per tail-beat cycle with only a small increase in the lost kinetic energy in the wake, suggesting that the resonant swimmers are more efficient.

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I. INTRODUCTION

When fish swim, their bodies bend because of internal muscle forces, but also because of external forces from the environment. These two forces are coupled together by the stiffness, damping, and other passive mechanical properties of the fish's body. The motion that we observe is a delicate balance of these internal and external forces, filtered by the body mechanics [1]. Some fish are stiffer than others [2]; some fish have more internal damping than others [3, 4]. The impact of these differences in material properties is not known.

Even though the body's material properties most likely have a strong effect on swimming performance, it is extremely difficult to use animal experiments to identify its role. While one species of fish may be stiffer than another, they also typically differ in numerous other ways, such as the anatomy of the muscle and skeleton and the way they activate their muscles during swimming. Instead, computational or robotic models offer a more controlled way to separate the different contributions of muscle activation patterns, body mechanics, and overall anatomy [*e.g.,* 5, 6].

Our recent computational simulations of lamprey swimming [7, 8] showed that stiffer swimmers accelerate faster but use more energy than more flexible swimmers, as long as the peak muscle force increases or decreases to match the body stiffness. Fluid-mechanical resonance may explain some of these differences. Quinn *et al.* [9] and Alben [10] found that flexible flapping or undulating panels can have multiple peaks in thrust or efficiency that depend on a set of nonlinear resonant interactions between the fluid and the body. In our previous work [11], by changing the stiffness, but keeping the activation frequency constant, we were changing how close the system was to one of the resonant interactions.

In that study, we did not examine the fluid dynamic mechanisms underlying the performance differences. How does the wake structure correlate with swimming performance? When an animal produces a more coherent wake, we expect that it should swim more efficiently [12], but at the same time, differently organized wakes can result in similar swimming efficiency [13]. We might expect there to be an optimal stiffness for producing a coherent wake. Animals with very stiff bodies may be able to support a high velocity difference from one side of their body to the other, so that they could shed a strong shear layer into their wake; shear layers tend to be unstable [14], so such a stiff swimmer might produce an incoherent wake with many extra vortices. At the other end, very flexible bodies may deform

too much during motion so that they do not shed individual, concentrated vortices [6]. At some intermediate stiffness, an animal might be able to produce an optimal coherent wake. Because the swimming is a resonant interaction with the fluid, though, the wake will also depend on the oscillation frequency.

The wake may also depend on the pressure distribution along the body. Fishes like eels and lampreys, which swim in an anguilliform mode with about one complete undulatory wave on their body [15], will have multiple regions of high and low pressure along their bodies [16, 17]. These changes in pressure tend to correlate with a 2P wake structure [18], in which the swimmer sheds two pairs of vortices in each full tail beat [6, 19]. Fishes like sunfish, which swim in a carangiform mode with much less than a full undulatory wave, will tend to have fewer changes in pressure along their bodies [20, 21]. During steady forward swimming, such fishes generally produce 2S wakes, with two single vortices shed each cycle [15]. Here, we consider how the pressure fluctuations near the tail correspond to vortex shedding into the wake.

Both the computational and robotic swimmers that we examine are extremely flexible, compared to previous studies of flexible foils. There are not many measurements of the stiffness of fish bodies, but Long [22] measured eels to have a passive stiffness (Young's modulus, E) as low as $0.39\,\mathrm{MPa}$, corresponding to a bending modulus EI of $1.6 \times 10^{-4}\,\mathrm{N\,m^2}$. Based on our experience, lamprey bodies are even more flexible than those of eels. Most previous studies of flexible propulsors, though, have used thin plastic foils with stiffness in the GPa range, which, because of their thinness, have bending moduli close to those of fish. For example, the most flexible panel in [9] had a modulus $EI = 0.6 \times 10^{-4}\,\mathrm{N\,m^2}$, but $E = 3.8\,\mathrm{GPa}$. To our knowledge, our study is the first to examine flexible propulsors that have material properties similar to those of fish and also match the relative thickness of the body.

To study these effects, we compare flow patterns from Leftwich *et al.* [6] around a robotic anguilliform swimmer with a flexible tail to those from Tytell *et al.* [7, 11] around a two-dimensional computational simulation of a flexible anguilliform swimmer. Both models are simplifications. The robotic swimmer has a passively flexible tail, while animal's muscles actively deform their bodies. The computational swimmer has active segments that approximate muscle running all the way down to its tail, like animals, but it is has only two spatial dimensions, while animals clearly have three. Nevertheless, we show that both models accu-

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rately capture important features of anguilliform swimming and serve as useful ways to study the interaction of material properties, muscle activation, and wake structure for swimming. Moreover, numerical convergence studies presented below show that high resolution detail of vortex formation in the computational algorithm is not necessary to accurately resolve the coupled fluid-body interaction. Based on these robotic and computational models, we investigate the role of flexibility on both wake formation and swimming performance.

II.   METHODS

A.   Methods for simulations

Tytell *et al.* [7] completed two-dimensional simulations of anguilliform swimming using an adaptive mesh immersed boundary method [IBAMR; 23, 24]. These simulations differ from much previous work in that the motion of the swimmer is not prescribed. The swimming motion instead is the result of the interaction between the fluid, described by the Navier-Stokes equations, and a body model that describes both the passive elasticity and the active muscle contraction for a lamprey. The body is constructed from three filaments (a "backbone" and left and right sides), which are connected by a network of elastic filaments that approximate the geometry of serial blocks of muscle in fishes, called myomeres [25]. Internal filaments are Hookean springs, and resist both compression and extension. Filaments along the left and right sides only resist extension, like collagen fibers. Additionally, left and right side segments produce active forces, which are described by a Hill-type muscle model, after Williams *et al.* [26], Hamlet *et al.* [27].

Body stiffness was altered by changing the stiffness of the passive springs that connect the filaments. An effective Young's modulus was estimated by bending the body through a set of known angles and calculating the energy stored in the springs [28]. At the same time, the maximum muscle force was increased or decreased to match the change in body stiffness; this produced swimmers of different stiffness that swam with similar tail beat amplitude.

We prescribed a traveling wave of activation with a defined period (usually 1 s) and one full wave on the body. This wave activated the muscle, which produced force, bending the body and interacting with the fluid. The swimmer was started from rest with a straight body and accelerated until it reaches a steady swimming speed.
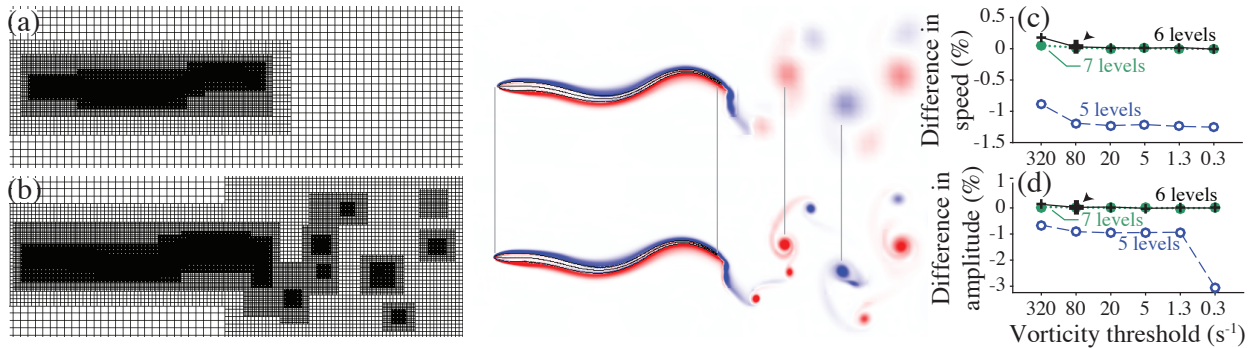
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Figure 1: Convergence tests of the adaptive mesh algorithm. Computational meshes and vorticity for (a) 5 refinement levels, vorticity threshold $320\,\text{s}^{-1}$, (b) 6 refinement levels, vorticity threshold $80\,\text{s}^{-1}$. Percent differences in swimming speed (c) and tail amplitude (d) for different vorticity thresholds and levels of refinement. *[2 column width.]*

1. Convergence testing

We checked to make sure that the parameters of the adaptive meshing algorithm did not affect the computed motion of the swimmer. The overall motion was very robust to large changes in these parameters. This algorithm [23] adaptively increases the grid resolutions in areas with high vorticity or close to the material points of the swimmer. Two parameters control the adaptive mesh. First, the resolution doubles each time the vorticity changes by greater than (1) a vorticity threshold parameter, up to (2) a maximum number of refinement steps. We tested how these two adaptive mesh parameters would affect the overall motion of the swimmer.

Fig. 1 shows the results of testing with three different maximum grid refinement levels (5, 6, or 7 levels) and six thresholds for vorticity (from $5\,\text{s}^{-1}$ to $320\,\text{s}^{-1}$). The higher the vorticity threshold, the less often the resolution will increase. At the lowest refinement and the highest vorticity threshold, the adaptively refined Cartesian grid had on the order of 5×10^4 grid cells, whereas at the highest refinement and tightest vorticity threshold, the number of grid cells was approximately 5×10^6. Although the wake in Fig. 1a is very smoothed and poorly resolved, the body shape and swimming velocity is not very much different from the more highly resolved case (Fig. 1b). Additionally, as long as there were at least 6 levels of refinement, there were negligible differences in the emergent properties of the coupled fluid-body interaction, such as the overall swimming speed (Fig. 1c) and tail

¹³⁴ beat amplitude (Fig. 1d). For that reason, all of the simulations here we performed with
¹³⁵ these intermediate parameters (indicated by an arrowhead in Fig. 1c and d).

B. Methods for robotic experiments

¹³⁷ The robotic platform used in this work is a modification of that used by Hultmark *et al.*
¹³⁸ [29], and the same as that used by Leftwich and Smits [16] and Leftwich *et al.* [6]. It
¹³⁹ consists of 11 servo motors attached in alternating pairs. Each motor is controlled by a
¹⁴⁰ BasicX Stamp micro controller (Parallax, Inc., Rocklin, CA, USA) that is programmed
¹⁴¹ to replicate the growing, traveling sine wave of anguilliform swimming. The motors and
¹⁴² controller for the robot are shown in Fig. 3(c) at the top of the panel. The motion of the
¹⁴³ robot is based on the work of Tytell and Lauder [19] and detailed in Hultmark *et al.* [29].
¹⁴⁴ The motors and controller shown in Fig.3(c) were covered in a custom-made latex skin to
¹⁴⁵ isolate them from the water. Swimming speed and flow data were taken with three passively
¹⁴⁶ flexible tails ($E = 0.12\,\mathrm{MPa}$, $0.17\,\mathrm{MPa}$ and $0.23\,\mathrm{MPa}$, corresponding to bending moduli EI
¹⁴⁷ $= 1.0$, 1.4, and $1.9 \times 10^{-4}\,\mathrm{N\,m^2}$). These tails are geometrically similar to the silver lamprey
¹⁴⁸ (*Ichthyomyzon unicuspis*) and cast out of flexible PVC gel (M-F Manufacturing Co., Fort
¹⁴⁹ Worth, TX, USA). A plastic insert in the center of the flexible tail controls the degree of
¹⁵⁰ flexibility of each tail.

¹⁵¹ The experiments were conducted in a closed loop, free surface water channel with a test
¹⁵² section that is $0.46\,\mathrm{m}$ wide, $0.3\,\mathrm{m}$ deep, and $2.5\,\mathrm{m}$ long. One honeycomb and three screens sit
¹⁵³ upstream of the 5:1 contraction. The anterior part of the robot was held in the test section
¹⁵⁴ at mid-depth by a frame mounted on an air-bearing sled free to move in the streamwise
¹⁵⁵ direction. Waves were eliminated by mounting a clear acrylic plate at the surface. Free
¹⁵⁶ swimming speeds were determined based on the water speed in the tunnel when the robot
¹⁵⁷ did not move in the streamwise direction while activated.

¹⁵⁸ The flowfield generated by the steadily swimming lamprey was measured in the horizontal
¹⁵⁹ mid plane of its wake using particle image velocimetry (PIV). The flow was seeded with silver
¹⁶⁰ coated hollow ceramic spheres with a diameter of $100\,\mathrm{\mu m}$ and an average specific gravity of
¹⁶¹ 1.01 (Potters Industries Inc. Conduct-O-Fil AGSL-150-30 TRD). A Spectra Physics 2020
¹⁶² Argon laser, wavelength of $490\,\mathrm{nm}$ (Newport Corporation, Mountain View, CA, USA), was
¹⁶³ then used to create a light sheet with an optical fiber delivery system and a Powell lens

164 (Oz Optics Ltd, Ottawa, ON, Canada). The sheet thickness was typically 1.5 mm ($1/e$
165 thickness). The sheet was oriented parallel to the robot, in the mid-plane of the swimming
166 robot. A Redlake MotionXtra HG-LE (IDT, Tallahassee, FL, USA) camera was mounted
167 perpendicular to the laser sheet (above the water channel) and used to capture the image
168 pairs with 8 to 10 ms between images. Pairs were taken at 10 Hz. Image pairs were captured
169 with a time delay of 8 to 11 ms between images. Ten image pairs were captured each second.
170 Exposure times were typically 3 ms. The acquired image pairs were processed using an
171 in-house PIV code (details of the code are presented in [30]). Interrogation windows of 32
172 pixels were used with 50% overlap between windows.

173   In addition to the velocity field, the fluctuating pressure signal on the surface of the
174 swimming body and an actively bending, but otherwise rigid tail was measured using a
175 Validyne DP 15 (Validyne Engineering Corp in Northridge, CA, USA) differential pressure
176 transducer. This was connected to a CD379 Validyne carrier demodulator, with a range of
177 0.85 to 1.4 kPa and an accuracy of ±0.05% of the full scale deflection. Ten pressure ports
178 were placed on the robotic lamprey. Four were located in the rigid part of the active tail,
179 and six were imbedded into the latex skin that surrounds the robot. The transducer was
180 connected to a NI USB-6212 data acquisition board (National Instruments, Austin, TX,
181 USA) and computer.

182   **C. Methods for data analysis**

183   *1. Scaling flexibility*

184   The flexibility of the computational swimmer and the robot were scaled according to the
185 analysis in Quinn *et al.* [9] to produce a nondimensional effective flexibility

$$\Pi_1 = \left(\frac{\rho h l^5 f^2}{EI} \right)^{1/2} \tag{1}$$

186 where ρ is fluid density, h is the average height of the swimmer, l is the length, EI is the
187 bending modulus, and f is the tail beat frequency. For the 2D computational swimmer, we
188 assume that h is equal to the average width.

2. Measuring the resonant frequency

For both robotic and computational models, we identified the mechanical resonant frequency of several of the models by oscillating them up and down in water and measuring the frequency that produced the highest amplitude motion. For the robot, this produced a clear resonant peak at 0.28 Hz for the 0.17 MPa tail. For the computational swimmer, the resonance was more complicated, with peaks at least at 0.5 Hz and 1.5 Hz; see the discussion in Tytell *et al.* [11]. However, we observed a clear maximum in tailbeat amplitude for the freely swimming model at 0.5 Hz and so we refer to this as the resonant frequency [11].

3. Material properties of the swimmers

Throughout this paper, we are comparing a robot with three different passive tails, one of which was also oscillated at its resonant frequency (Table I). The body anterior to the tail was moved actively using servomotors that matched a given waveform. The tails were shaped to match the shape of a lamprey's tail, and were 20% of the total length of the robot (0.9 m). See Hultmark *et al.* [31] and Leftwich *et al.* [6] for more details on the robot. Similarly, we compare the computational swimmer with five different passive stiffnesses, one of which was oscillated at the resonant frequency (Table I). Different from the robot, in the computational model, we adjusted the underlying material properties, but kept active force generation along most of the entire length. See Tytell *et al.* [7] and Tytell *et al.* [11] for more details on the computational swimmer.

4. Phase averaging for experimental measurements

The velocity data from the PIV experiment and the pressure data, both obtained with the robotic lamprey, were phase averaged over 20 cycles of motion. For the PIV data, 10 velocity fields were measured per second. The robot was programmed to have a period of motion of exactly 1.8 s per cycle. Therefore, images at the same cycle of motion (i.e images, 1, 19, 37, 55, *etc.*) were averaged to produce 18 velocity fields representing the entire cycle of motion. The same method was employed to determine the average pressure field throughout the cycle. However, the temporal resolution of the pressure transducer was significantly higher (1 kHz). Again, 20 cycles of motion was used to calculate the average

	E (MPa)	EI ($\times 10^{-5}\,\mathrm{N\,m^2}$)	Frequency (Hz)	Π_1
Robot	0.12	10	0.56	3.10
	0.17	14	0.56	3.61
	0.23	19	0.56	4.27
	0.17	14	0.28	1.81
CFD	0.59	0.21	1.0	6.63
	0.64	0.23	1.0	6.36
	0.76	0.26	1.0	5.87
	0.98	0.34	1.0	5.15
	1.06	0.37	1.0	4.96
	1.06	0.37	0.5	2.49

Table I: Parameters for robotic and computational experiments

pressure cycle.

5. Power calculations

We calculated a measure of efficiency for each swimmer by estimating the kinetic energy flux $\dot{K}$ (a measure of the wake power):

$$\dot{K} = \rho h \int_C \mathbf{u}^2\, \mathbf{u} \cdot \mathbf{n}\, ds \tag{2}$$

where ρ is the density of the fluid, h is the vertical height of the swimmer, $\mathbf{u}$ is the fluid velocity, $\mathbf{n}$ is the unit vector normal to the edge of the planar control area C, ds is the distance along the edge of the control area. The wake power was scaled for each swimmer to produce a power coefficient C_K [32, 33] by dividing by the power required to overcome drag:

$$C_K = \frac{\dot{K}}{0.5\rho S U^3}, \tag{3}$$

where S is the wetted surface area of the swimmer, and U is the mean swimming speed. Error on C_K was calculated for experimental measurements based on the standard deviation
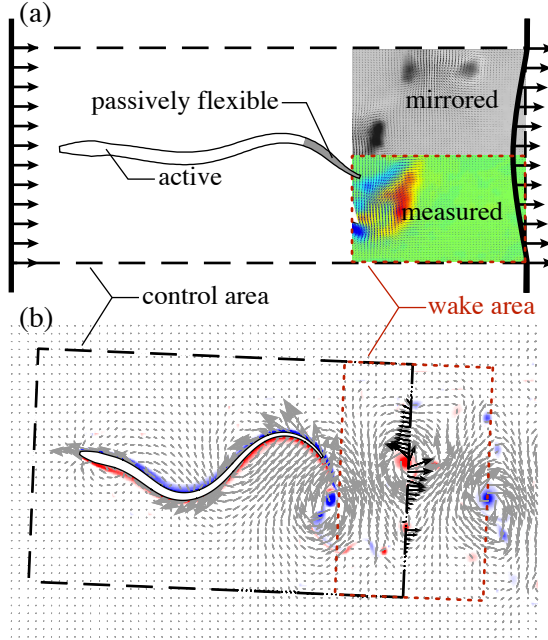
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Figure 2: Example control areas for the swimmers. (a) Robot. (b) Computational simulation. The boundary of the control area is given by a black dashed line, and the wake region for turbulence estimation is outlined with a red dashed line. *[1 column width.]*

228 of the PIV measurements [34]

229   The $\dot{K}$ is the flux through a control volume containing the entire robot. The incoming
230 velocity was fixed for each experiment, and was assumed to be uniform. While the PIV data
231 only encompasses the left side of the wake, the full wake can be constructed by mirroring
232 the data and offsetting it by one half of a cycle (labeled "mirrored" in Fig. 2a). In this
233 analysis, we neglect the contributions through the sides of the control volume upstream of
234 the tail, for which there is no velocity data.

235   For the computational swimmer, we defined an area (Fig. 2b) surrounding the swimmer,
236 moving at a constant velocity $\bar{\mathbf{U}}$ equal to the mean swimming speed and direction. Because
237 of initial transients, the swimming direction was often angled relative to the horizontal axis,
238 but the control area was aligned to the swimmer, as shown in Fig. 2b. Fluid velocity was
239 interpolated from the grid to the contour surrounding the area and integrated to estimate
240 thrust, lateral forces, and wake power, all of which are defined per unit height.

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6.   Turbulence parameters

To assess the coherence and regularity of the wake, we estimated a phase-dependent turbulence intensity of the wake. First, we phase averaged the wake at 18 different phase values over at least three cycles and then computed the relative mean squared difference of the instantaneous fields with the phase averages:

$$I = \frac{1}{2} \left\langle \frac{(u - \overline{u_\phi})^2 + (v - \overline{v_\phi})^2}{\overline{u_\phi}^2 + \overline{v_\phi}^2} \right\rangle \tag{4}$$

where $\overline{u_\phi}$ and $\overline{v_\phi}$ are the average mean components of velocity at a particular location and phase ϕ, and $\langle \cdot \rangle$ denotes an average over both space and phase. The value of I does depend on the number of cycles averaged, but the pattern relative to flexibility was very robust. Note that (u, v) includes the mean swimming speed U, even though it has been subtracted in figures showing wake flow patterns. For the robot, we used the measured wake region, which included half the wake laterally and approximately one spatial period behind the tail (Fig. 2a). Error on I was calculated for experimental measurements based on the error of the PIV measurements [34] For the computational swimmer, we used a domain in the wake from just behind the mean tail position to one full wake cycle ($= Uf$, where U is the mean swimming speed and f is the tail beat frequency) downstream (Fig. 2b).

III.   RESULTS AND DISCUSSION

A.   Wake structures of the robot, the simulations, and living eels

All of the swimmers shed a wake with two pairs of primary vortices shed per cycle, and the basic structures match qualitatively (Fig. 3). In both the wake of the the living eel (swimming at $1.5\,\text{lengths}\,\text{s}^{-1}$; [19]) and robotic swimmer [31] (shown in Fig. 3a and c respectively) a strong, repeatable pair of similar strength vortices was shed each half cycle— a two pair (2P) wake structure [35]. The underlying wake structure for the computational swimmer is also a 2P structure [7]. However, in Fig. 3(b) we see that the primary vortex (1) often splits a second time (1a). Thus, each cycle the computational swimmer produces a primary and secondary vortex of similar strength, as well as a weak tertiary vortex. In no case did any of the swimmers produce a 2S wake, although the wake of accelerating eels becomes closer to a 2S pattern [36]. Previous studies have observed that flapping propulsors
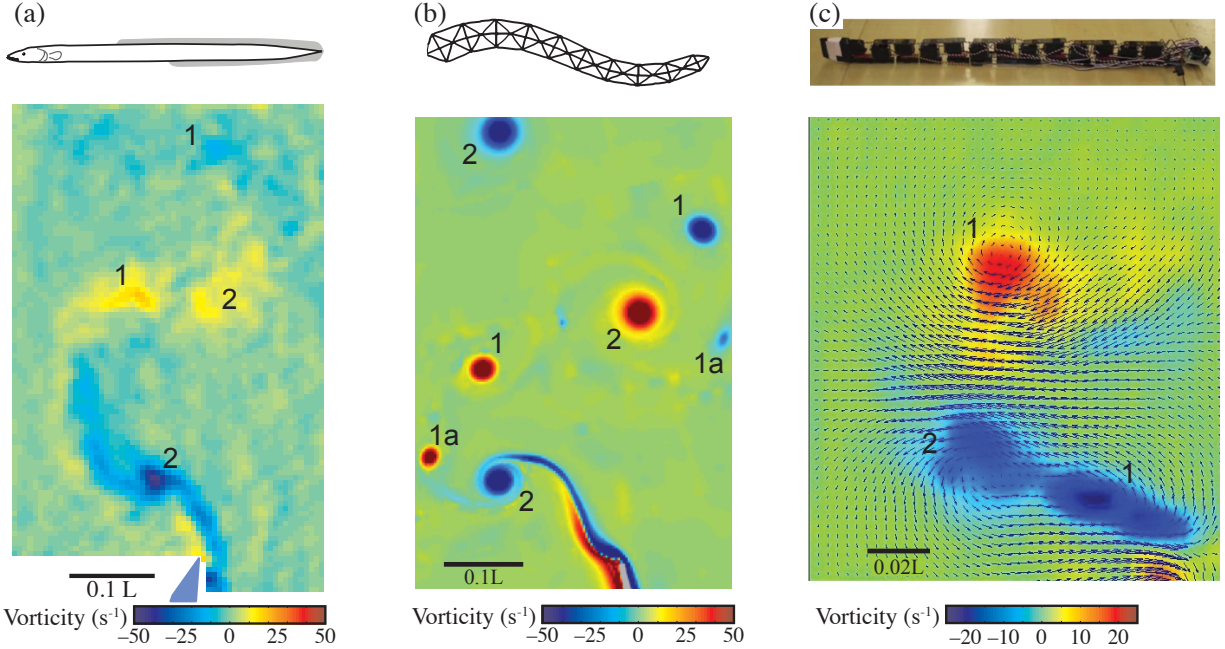
Figure 3: Wakes of a swimming eel (a), the computational swimmer (b), and the robotic swimmer (c) all show the same overall 2P wake structure. The tails of the swimmers are at the bottom of each panel. Data replotted from [7, 19, 31].

tend to produce 2P wakes when Strouhal number St is high [37, 38], as it is for our swimmers ($St \geq 0.5$ for all cases). Using a more fish-like geometry, Borazjani and Sotiropoulos [39] also found that lamprey shaped swimmers almost always produced 2P wakes, except when the Strouhal number was 0.2 or less.

Both the computational and robotic swimmers that generated the data presented in Fig. 3(b) and (c) had active tails. The muscles or motors extended to the end of the swimming body. The coherence and repeatability of the wake degraded as the tail became more passive. Additionally, the presence of higher order wake patterns was more prominent in these cases. While the main structure generally remained a 2P wake, it was common to see distinct structures in addition to the primary and secondary pair. These structures were neither repeatable not predictable from cycle to cycle, but may contribute to the increase in the wake power for more flexible swimmers.
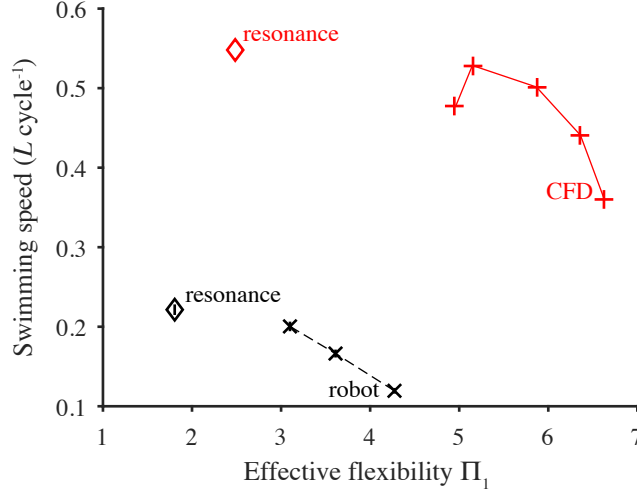
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Figure 4: Normalized swimming speed for robotic (black) and computational (red) swimmers of different stiffness, when oscillating at a constant frequency (dash lines) and at a resonant frequency (diamonds). Error bars may be smaller than symbols. *[1 column width.]*

B. Very flexible swimmers swim slower, but resonance increases swimming speed

For very flexible swimmers (Young's modulus $E < 1\,\mathrm{MPa}$, which included all of the robotic swimmers and all but the least flexible computational swimmer; Table I), both computational and robotic, decreasing stiffness leads to decreasing swimming speed (Fig. 4). For both types of swimmers, we adjusted the passive stiffness and then tuned the muscle force or motor power so that the overall tail beat amplitude remained approximately the same. For the computational swimmer, doubling (or halving) the peak muscle force to match a doubling (or halving) of the body stiffness tended to preserve the overall amplitude [7].   For comparison, eels have a stiffness of approximately $0.39\,\mathrm{MPa}$ [22]. Under these conditions, increasing flexibility leads to decreasing swimming speed (Fig. 4). When the motion is determined by a balance of fluid forces and internal forces, as it is for the computational swimmer, there is an optimal stiffness for maximum swimming speed (Fig. 4b) at the same tail beat frequency. For flexible bodies, like the robot tail, fluid structure interactions can deform the body, reducing the transfer of momentum from the body to the water and lowering the swimming speed.

If the tail beat frequency changes, mechanical resonance can improve the swimming speed. Oscillating at the resonant frequency decreases the effective flexibility (see Eq. 1)

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₂₉₇ and produces dramatic increases in the distance traveled per tail beat cycle (red diamonds
₂₉₈ in Fig. 4). For both types of swimmer, the resonant frequency was lower than the frequency
₂₉₉ used in most of the tests; this means that the absolute swimming speed in $\mathrm{m\,s^{-1}}$ is lower for
₃₀₀ the resonant swimmers. But in each case, swimming at the resonant frequency is much more
₃₀₁ effective, traveling much further during each tail beat than the non-resonant swimmers.

₃₀₂ Quinn *et al.* [9] found a complex relationship between thrust force and stiffness for teth-
₃₀₃ ered flexible panels when they heave from side to side. They measured force and propulsive
₃₀₄ efficiency for foils with different stiffness, as they varied the flapping frequency and the
₃₀₅ oncoming flow speed. They found multiple distinct resonant peaks in thrust force as fre-
₃₀₆ quency varied. In general, however, at a given frequency, they found that more flexible
₃₀₇ foils produced lower forces than stiffer foils [9], matching our results. Similarly, numerical
₃₀₈ simulations of freely swimming foils in an inviscid fluid showed resonant peaks in swimming
₃₀₉ speed, but with speed decreasing on average as flexibility increased [40].

₃₁₀ The mechanical resonant frequency of a flapping foil in water is not necessarily the fre-
₃₁₁ quency that produces the fastest swimming speed [41], but often the optimal frequency is
₃₁₂ close to the mechanical resonance. When both the robot and the computational swimmer
₃₁₃ used frequencies above the resonant frequency, they swam slower than when they used the
₃₁₄ resonant frequency. It is possible that even lower frequencies might have produced faster
₃₁₅ swimming, but we did not test those frequencies.

C. Pressure increases near the tail as vortices are shed

₃₁₇ The fluctuations in pressure near the tail correspond to the timing of vortex shedding.
₃₁₈ Fig. 5 shows the pressure coefficient C_P as a function of time for two full tail beat cycles for
₃₁₉ both the robotic and computational swimmer. The pressure coefficient is defined as

$$C_p = \frac{p}{\frac{1}{2}\rho U^2} \tag{5}$$

₃₂₀ where p is the pressure measured from the transducer as discussed in section II B. In both
₃₂₁ cases, the pressure value used was taken from $0.98L$ from the head of the swimmer (very
₃₂₂ near the tip of the tail).

₃₂₃ When C_P hits a local maximum (labeled with numbers), vortices are shed into the wake
₃₂₄ (right side). The global peak, labeled '1' in both Fig. 5a and b comes just before the tail

₁₅

changes direction and begins moving away from the pressure port. This is just after the maximum acceleration of the tail at that location. The pressure takes over half the cycle (82% for the robotic swimmer and 60% for the computational swimmer) to reach its global minimum.

Using the velocity fields measured or computed, we can compare the pressure signal to the coherent structures present near the swimming surface. In Fig. 5 the panels on the right show the vorticity near the tail at the times marked in the figure. Panel (1) shows the velocity field at the global maximum of the pressure. We see that the large starting-stopping vortex that will form the primary vortex 1a from this half cycle has just detached from the tail. As this happens, the pressure rises dramatically, just before the peak labeled '1'. This point occurs when the tail has just passed the midpoint of the robot and is moving to the left (indicated by the gray arrow). Point '2' is the second maximum—occurring just after the tail reaches its maximum amplitude and has changed directions. At this time, the pressure has decreased somewhat as the local boundary layer has weakened. These spatial and temporal fluctuations in the pressure gradients then change the local freestream velocity, which changes the local circulation. Ultimately, the time varying circulation establishes the strength of the trailing vortices shed into the wake.

D. Very flexible tails produce less repeatable wakes

For very flexible swimmers, increasing flexibility leads to increasingly disorganized wakes (Fig. 6, 7). The phase-dependent turbulence intensity (Fig. 8) shows that, at a constant frequency, the wakes become less organized as flexibility increases, at least for very flexible swimmers ($E < 1\,\mathrm{MPa}$). Regardless of flexibility, the robot (Fig. 6) always produces a 2P wake, with two pairs of coherent structures that largely contain opposite signed vorticity. As the flexibility increases, the robot's wakes become far more chaotic: the structures have less concentrated vorticity and more often have patches of oppositely signed vorticity. This is seen in the differences between panels (a) and (b) of Fig. 6. While in both cases, a positive and negative vortex can be identified, the coherent, oppositely signed pair—the hallmark of the 2P wake— is clearly visible immediately downstream of the body for the moderately stiff tail (panel b). The very flexible tail (panel a) produces a pair that is of unequal strength and inconsistent location in the wake.
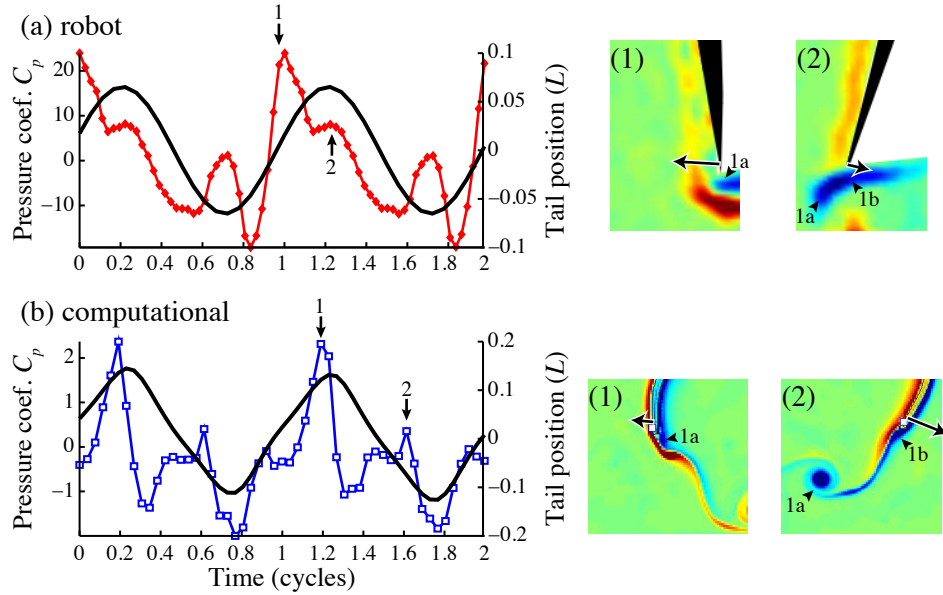
Figure 5: Pressure traces and flow fields near the tail for the robotic (a) and computational (b) swimmers. Panels on the right show example wakes at the times labeled with arrows and the approximate tail velocity is shown with a gray arrow. *[1.5 column width.]*

For the computational swimmer (Fig. 7), very flexible swimmers also produce less coherent wakes, but the pattern is different: rather than shedding disorganized structures, the computational swimmer produces more vortices of alternating sign that are less repeatable (Fig. 7a). For example, in Fig. 7a, note the presence of small packets of vorticity near the vortex at the top of the panel; these do not tend to line up well with the black contours from the previous cycle. As in the case of the robot, the wake of the computational swimmer begins to regain coherence as the tail is stiffened (Fig. 7b and c). Fewer structures are found in each cycle, and the vortex locations are more consistent from cycle to cycle. This can be seen in the greater overlap between the colored vortices and the black outlines in Fig. 7c, compared to Fig. 7a particularly. Because the computational swimmer is 2D, it almost always produces a shear layer off the tip of the tail that then rolls up into additional vortices in the wake. An equivalent 3D swimmer would most likely produce a less intense shear layer, because flow could move vertically around the tail, rather than just being shed off in the horizontal plane.

Phase-dependent turbulence intensity follows the patterns seen in Fig. 6 and 7. For the robot, the swimmer with the stiffest tail has the lowest turbulence intensity and the highest
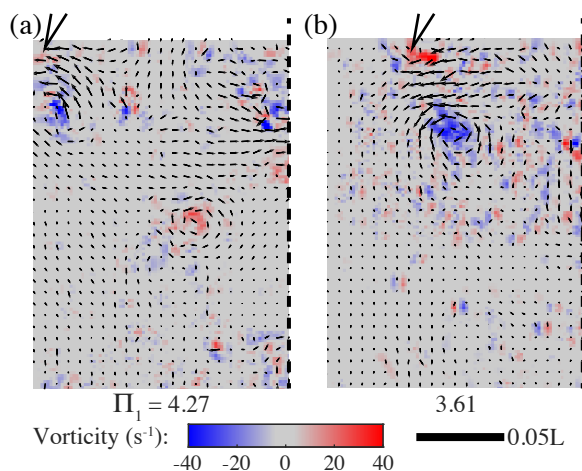
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Figure 6: Flow patterns in the wake of the robotic lamprey with a very flexible tail (a) and a moderately stiff tail (b). The approximate position of the tail is shown at the top and the dashed line indicates the middle plane of symmetry [1 column width.]
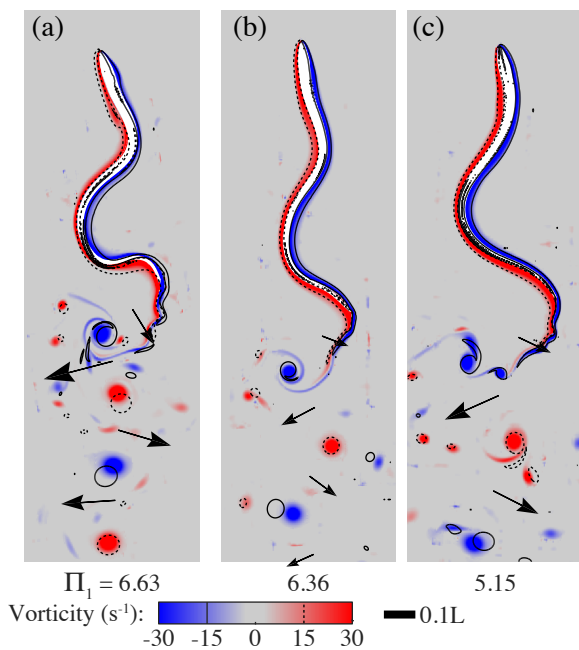

Figure 7: Vorticity in the wake of the computational swimmer with a very flexible body (a), a moderately flexible body (b), and a relatively stiff body (c). Vorticity in one cycle is shown in color, while vorticity contours from the previous cycle are overlaid in black. [1 column width.]
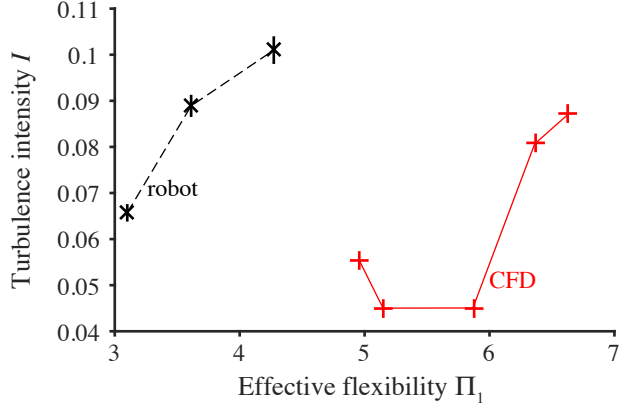
Figure 8: Phase-dependent turbulence intensity for the robot (black) and computational swimmer (red). *[1 column width.]*

swimming speed (Fig. 8a, 4a). For the simulation, the intermediate stiffnesses have the lowest turbulence intensity, but the stiffer swimmer, which has a somewhat less repeatable wake, swims faster (Fig. 8b, 4b). Resonance seems to increase the regularity of the wake, particularly for the robot, but since the resonant frequencies were much lower than the test frequencies shown in Fig. 8, the turbulence intensity results from resonant trials are not directly comparable and are therefore not shown.

For the computational swimmer, we were able to examine how the changes in presssure near the tail relate to the stiffness of the body and the coherence of the wake. Fig. 9 shows that, for the two most flexible swimmers, the pressure varies over a higher range than the stiffer swimmers ($E > 0.76\,\mathrm{MPa}$). For the stiffest swimmer ($E = 1.06\,\mathrm{MPa}$), the pressure tends to have small jumps, while at intermediate stiffness, the pressure varies both smoothly and over a smaller range. These swimmers also swim fastest (Fig. 4b) and have wakes with lower turbulence intensity (Fig. 8). At an intermediate stiffness (0.98 MPa) we find that the pressure fluctuates the least. The intermediate swimmer also has the most coherent wake (Fig. 7b, Fig. 8). The larger fluctuations in pressure for the high and low stiffness swimmers corresponds to increased vortex shedding and a less coherent wake (Fig. 7a and c).

E. Wake power

We find that more flexible swimmers dump relatively more kinetic energy into the wake than stiffer swimmers, resulting in a larger wake power coefficient (Fig. 10). This repre-
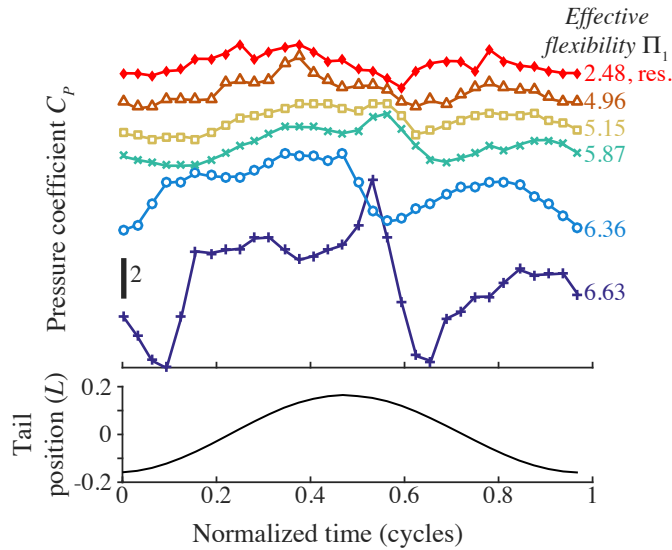
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Figure 9: Pressure coefficient near the tail tip and tail position for computational simulations with different stiffnesses. For visibility, the pressure traces have been offset from one another, arranged from the most flexible on the bottom to the stiffest on the top. The top (red) trace shows pressure from the $0.5\,\mathrm{Hz}$ swimmer, which is at its resonant frequency. *[1 column width.]*
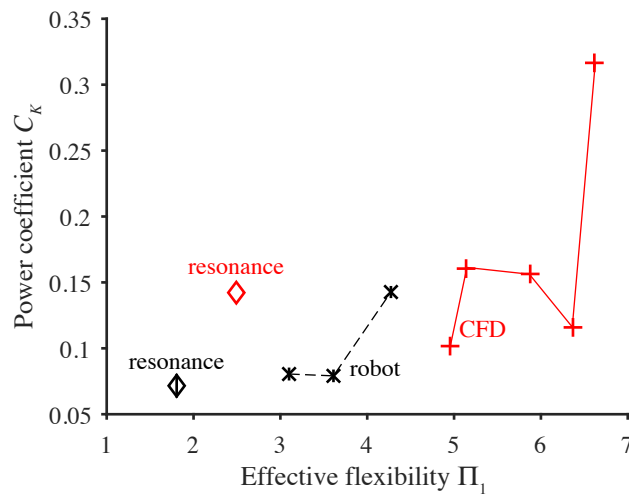

Figure 10: Wake power coefficients for the robotic (black) and computational (red) swimmers with different effective flexibility. *[2 column width.]*

390 sents wasted energy, the portion of the overall energetic budget that does not help propel
391 the animal forward, indicating that the more flexible swimmers are probably less efficient
392 than the stiffer swimmers.   Efficiency is difficult to assess during steady free swimming

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[32, 33, 42]. The total force on a swimmer, averaged over a cycle period, must be zero, because the swimmer is neither accelerating nor decelerating. For anguilliform swimmers like those studied here, the thrust and drag forces also balance fairly evenly along the body [42]. Thus, neither thrust nor drag can be measured directly, even for the computational swimmer. Various groups [43, 44] have used different approximations for thrust or drag. In particular, Raspa *et al.* [44] found that thrust from elongated body theory matched the drag due to streamwise vortices shed along the top and bottom of a flapping plate. It is not clear, however, how this would apply to a 2D swimmer, which cannot shed these types of vortices. Instead, we use the wake power, a different measure of efficiency. Higher wake power indicates a less efficient swimmer [42].

In a previous paper [6], we used elongated body theory to estimate the mean thrust for the robot with flexible tails. We found that the thrust decreased as flexibility increased. Here, we find that the wake power coefficient increases with increasing flexibility (Fig. 10); together, the decrease in thrust and increase in wasted power show that efficiency drops dramatically as flexibility increases, for these very flexible swimmers. Other groups [e.g. 45, 46] have found that a passively flexible tail or trailing edge flap can increase swimming performance, but these studies used much stiffer materials.

The increase in wake power is largely due to an increase in the lateral velocities in the wake. Fig. 6 and 7 show the wakes for the computational and robotic swimmers, with representative velocity vectors show in the jet regions of the wake. Note that the lateral velocities are higher for the more flexible swimmers (Fig. 6a and 7a, particularly), while the axial velocities are fairly similar.

Quinn *et al.* [9] found that maximum efficiency at resonant frequencies increased for flapping foils as they became more flexible, while we found that more flexible foils were less efficient. However, these results do not contradict ours, because the relationship was quite complex; depending on the flow speed and frequency, the efficiency could increase or decrease as stiffness changed [9].

IV.   CONCLUSIONS

Ton understand the role of body stiffness in the swimming performance of fish, it is very challenging to study actual fish. Related species may differ in stiffness [2], but they often

have many other differences. Instead, robotic and computational models offer a controlled way to analyze flow features and mechanics as a function of body flexibility [47]. While the models presented here are great simplifications of the true animal, they are not simple. The robotic model undulates its body to produce thrust, but its tail flexes passively in response to fluid forces. These interactions are still poorly understood. The computational model couples an actuated, elastic swimmer to the full Navier-Stokes equations, albeit in 2D. In order to characterize the bending modulus and the resonant frequency of the flexible components of each model, similar experiments that bent or oscillated the robot tail and the computational swimmer in a physical water tank or an *in-silico* tank were performed. Even though the 3D robot and the 2D computational swimmer are such different constructs chosen to represent an anguilliorm swimmer, we see excellent qualitative agreement when comparing many metrics. In both, an active swimmer produces a 2P wake structure, and increasing flexibility decreases the wake coherence and increases the instances of spontaneous vortex shedding. The coefficient of pressure very near the tip of the tail in both cases contains three local maxima per cycle. These maxima are connected to vortex shedding at the tail and vorticity fields show this connection. Not surprisingly, the wake features of the 3D robot match those of a swimming eel more closely than the 2D computational swimmer.

ACKNOWLEDGMENTS

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