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Broadband Cloaking of Flexural Waves

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The governing equation for elastic waves in flexural plates is not form invariant, and hence designing a cloak for such waves faces a major challenge. Here, we present the design of a perfect broadband cloak for flexural waves through the use of a nonlinear transformation in the region of the cloak, and by matching term-by-term the original and transformed equations and also assuming a pre-stressed material with body forces. For a readily achievable flexural cloak in a physical setting, we further present an approximate adoption of our perfect cloak under more restrictive physical constraints. Through direct simulation of the governing equations, we show that this cloak, as well, maintains a consistently high cloaking efficiency over a broad range of frequencies. The methodology developed here may be used for steering waves and designing cloaks in other physical systems with non form-invariant governing equations.

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I. INTRODUCTION

The method of *Transformation Optics*, which are based on transformation solutions [1] and originally developed in optics community for passive cloaking [2, 3], offers a novel method for controlling electromagnetic waves using the subtle idea of coordinate transformation. Based on this method, invisibility cloaks for electromagnetic waves were designed, fabricated and successfully tested [4, 5].

The most important necessary condition for applicability of the method of transformation optics is that the governing equations must be *form invariant* under coordinate transformation. Since physical systems admitting wave solutions share many common properties, it is well expected that the transformation optics method works in any wave system with form-invariant governing equation. This has been confirmed and cloaks have been designed and tested in a variety of other systems such as for acoustic waves [6–8], water waves [9–11] and matter waves [12].

Flexural waves, such as those propagating on a thin elastic plates, have a governing equation that is known to be not form-invariant [e.g. 13]. Therefore, the classical method of designing a cloak through transformation media method does not directly work in the context of flexural waves. While the method of transformation media cannot be directly applied, it has long been known that a thin light density region coating a denser inclusion reduces the shadow generated by a scatterer in low frequencies [14, 15], and therefore this thin lighter density coat can be used as a camouflage to reduce the scattered field at the far field in low frequencies. In order to expand this method to higher frequencies, a number of actively controlled sources should be added to the exterior of the object to decrease the scattered field at infinity [16, 17]. Note that these methods cannot be regarded as *invisibil-ity cloaks*, however they can reduce the leading monopole source term of a scatterer in the asymptotics at infinity [18] and therefore reduce the scattered field.

In order to use transformation optics method for the flexural waves, one crude approximation is to adopt a form-invariant equation whose form is close to the governing equation of flexural waves and then use classical linear cloak design [19, 20], which we refer to as *linear* because within the cloak, the radial direction is mapped with a function that is linearly dependent on the radius. While the resulting wave pattern about a to-be-cloaked cylinder may look like wave patterns of cloaking, a quantitative investigation of cloaking efficiency [21][22] reveals that such a cloak has a poor and in many cases even negative cloaking efficiency (i.e. an object with the cloak about it scatters even more energy than the object without one, see also Fig. 5, 6). Alternatively, if it is assumed that both density and elasticity of the material can be independently tuned, then a condition is obtained under which the highest order terms of the governing equation satisfy the cloaking requirement [18]. This is theoretically an improvement, as the highest order terms can be shown to play a more important role than the rest of the terms in the governing equation. Nevertheless, fabricating a material with a variable density and elasticity is a serious challenge [20]. Along the same line, more degrees of freedom such as several independent elastic parameters may be assumed to improve the theoretical performance [23], but this makes the realization of cloak in the physical space even farther from achievable.

Here we present the design of a perfect broadband cloak for flexural waves. For the cloak to be realizable in the physical domain, we put the constraint that the density ρ is constant and only the modulus of elasticity Ecan be changed across the cloak. We employ a nonlinear transformation for the cloaking region and, by choosing proper material properties with pre-stressing and body

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forces, match term-by-term the coefficients in the original and transformed equations. We show rigorously that the transformed equation matches perfectly with the prestressed orthotropic and inhomogeneous plate's equation with body forces. We then numerically test an experimentally feasible approximate adoptation of the cloak, and quantitatively analyze its effectiveness.

II. GOVERNING EQUATIONS

For an isotropic plate with thickness h and density ρ_0 , governing equation for out of the plane displacement $\eta(R, \Theta, t)$ in the z direction normal to the plate's surface reads [e.g. 24]

$$D_0 \Delta^2 \eta + \rho_0 h \eta_{tt} = 0, \qquad (1)$$

where $D_0 = E_0 h^3 / 12(1 - \nu^2)$ is the flexural rigidity, E_0 is the Young Modules, ν is the Poisson ratio, and Δ is the horizontal Laplacian operator in (R, Θ) directions.

To cloak a circular region A_c (radius a) with a cloak of outer radius b co-centered with A_c , we need to map the region of $0 \le R \le b$ to the cloaking region $a \le r \le b$. We use the nonlinear transformation \mathcal{F} defined as

$$\mathcal{F}: \begin{cases} r = \sqrt{(1 - a^2/b^2) R^2 + a^2}, \ 0 \le R \le b, \\ \theta = \Theta, \end{cases}$$
(2)

that has a special property of its Jacobian being a constant [11]. Using this transformation and further assuming a time-periodic motion of frequency ω , equation (1) is mapped to [using Lemma 2.1 in 25]

$$D_0 \tilde{\nabla}^2 \tilde{\nabla}^2 \eta - \rho_0 h \omega^2 \eta = 0, \qquad (3)$$

where

$$\tilde{\nabla}^2 = \left(1 - \frac{a^2}{b^2}\right) \left[\frac{1}{r}\frac{\partial}{\partial r}\left(\frac{r^2 - a^2}{r}\frac{\partial}{\partial r}\right) + \frac{1}{r^2 - a^2}\frac{\partial^2}{\partial \theta^2}\right].$$
(4)

Note that if a = 0, then $\tilde{\nabla}^2 \equiv \Delta$.

In a traditional cloak design for form-invariant governing equations [e.g. 2], material properties as functions of spatial variables are determined such that the transformed equation (3) with the new material properties becomes equivalent of the original equation (1). If we do the same here, the rigidity D becomes spatially variable in different directions, which means the required material for cloaking is *inhomogeneous* and *orthotropic*. The issue is, equation (1) with $D = D(r, \theta)$, is *not* the governing equation for an inhomogeneous and orthotropic plate. In fact, the governing equation for a general $D(r, \theta)$ is very much different in the look (see equation (A.11) in Appendix), and most importantly this equation is *not* form-invariant.

With this knowledge, we therefore look for material properties that result in the matching of the coefficients of the two equations (i.e. (3) and (A.11)). We find that if we choose the following material parameters

$$D_r = \alpha^2 \mathcal{A}^2(r) D_0, \tag{5a}$$

$$D_{\theta} = \alpha^2 \left(1/\mathcal{A}(r) \right)^2 D_0, \tag{5b}$$

$$D_{r\theta} = \alpha^2 D_0, \tag{5c}$$

$$\nu_{\theta} = \frac{1}{\alpha^2 \mathcal{A}^2(r)} \left[\mathcal{B}(r) - 4 \log \mathcal{A}(r) \right], \qquad (5d)$$

where $\mathcal{A}(r) = 1 - (a/r)^2$, $\alpha = 1 - (a/b)^2$ and $\mathcal{B}(r) = 3(r/a) \log [(r-a)/(r+a)] - 2a^2/(r^2 - a^2)$ then between equation (3) and (A.11) all terms that include 4th order derivatives (i.e. highest order appearing in these equation), all 3rd order, 2nd order and 1st order terms match perfectly except two extra terms in the transformed equation which are factors of $\partial^2 \eta / \partial r^2$ and $\partial \eta / \partial r$. Interestingly, these extra terms go to zero if the penetration depth is small [see Eq. (A.19) in Appendix]. Alternatively, these two extra terms can be handled with a material that is pre-stressed only in the radial direction (i.e. $N_{\theta\theta} = N_{r\theta} = 0$) with a radial body force (i.e. $S_{\theta} = 0$) according to

$$\frac{N_{rr}}{D_0} = \frac{1}{2a^2} \left[\left(\frac{a}{r}\right)^8 \mathcal{N}(r) + \frac{3}{2} \left(\frac{a}{r}\right) \log\left(\frac{r-a}{r+a}\right) \right], \quad (6a)$$

$$\frac{S_r}{D_0} = -\frac{3}{a^3} \left(\frac{a}{r}\right)^{11} \mathcal{S}(r) \tag{6b}$$

with $S(r) = \left[5 - 12(r/a)^2 + 8(r/a)^4\right] / \left[1 - (a/r)^2\right]^2$ and $\mathcal{N}(r) = \left[6 - 10(a/r)^2 - 2(r/a)^4 + 3(r/a)^6\right] / \left[1 - (a/r)^2\right].$

The above cloak for flexural waves is a rigorously derived *perfect* (i.e. efficiency is theoretically unity) and *broadband* cloak. We now move to the next level by designing an approximate adoption of this perfect cloak restricted by more physical constraints that make achieving such a cloak even easier in an experimental setting. Specifically, the goal is to find an approximate adoption of our cloak that only requires *concentric layers of homogeneous material* [c.f. 19, 20]. Using these concentric isotropic layers, we can only achieve a variable radial and azimuthal flexural rigidities.

Assuming only a variable radial and azimuthal rigidities in our cloak (i.e. equations (5a) and (5b)), only the highest derivatives (i.e. 4th order and 3rd order terms) match between equations (3) and (A.11). In order to test the effectiveness of our cloak which is based on the nonlinear transformation (2), in comparison with the one based on linear transformations [19, 20], we use N = 15 layers of homogeneous but orthotropic materials, that is, D_r, D_{θ} are constant throughout each layer to approximate (5a) and (5b). To achieve the required orthotropic response, each layer is divided into two sub-layers of isotropic and homogeneous materials with different rigidities to match (5a) and (5b) in that layer. These two sub-layers can be shown through homogenization that approximate the required orthotropic properties [e.g. 19, 20, 26, 27]. In implementing the cloak, since according to (5) the rigidities go unbounded at the inner boundary, a small offset



FIG. 1. Profile of the rigidity as a function of r required to achieve a cloak for flexural waves. Each layer is made up of a homogeneous and isotropic material, but the averaged properties provides an inhomogeneous and orthotropic apparent rigidity according to (5). In the design presented here, the number of layers is 15, each layer is divided to two sub-layers, and b/a = 4.

is introduced such that the rigidity of the first layer (next to r = a) is calculated at this offset distance from the inner boundary. This offset can be shown to be equivalent of transforming a region of $\varepsilon \leq R \leq b$ to the cloaking region $a \leq r \leq b$ [see Eq. (A.31) and (A.32) in Appendix]. In the numerical simulations that follow, we choose this offset to be 15% of thickness of a layer. The final profile of the rigidity of the isotropic and homogeneous layers for the case of b/a = 4 is shown in Fig. 1. We would like to note that the rigidity profile offered in Fig. 1 with a constant density profile is experimentally feasible: for example i. see table 1 in [19] where a list of polymers with 5 orders of magnitude changes in relative rigidities with a constant densities is offered or ii. see [20] where a combination of only two polymers, polyvinyl chloride (PVC) and polydimethylsiloxane (PDMS), provides us with a range of 3 orders of magnitude change in relative rigidities with a constant density.

III. NUMERICAL MODEL

In order to numerically solve the thin plate's equation with flexural rigidity as shown in Fig. 1, we use the spectral methods with Fourier expansion in the azimuthal direction and Bessel functions in the radial direction. Assuming time harmonicity of ω , the solution in each layer is therefore expressed as Re $[\eta(r, \theta) \exp(i\omega t)]$. We expand the spatial part $\eta(r, \theta)$ as

$$\eta(r,\theta) = \sum_{n=-\infty}^{\infty} \eta_n(r) \exp(in\theta), \qquad (7)$$

where $\eta_n(r) = A_n J_n(k_i r) + B_n I_n(k_i r) + C_n Y_n(k_i r) + E_n K_n(k_i r)$ with $k_i^4 = \rho_0 h \omega^2 / D_i$ and D_i being the flexural rigidity of the layer. Here, $J_n(.), Y_n(.)$ and $I_n(.), K_n(.)$ are respectively Bessel and modified Bessel functions of the first and second kind. Also A_n, B_n, C_n and E_n are

coefficients that are later found by satisfying the boundary conditions. These boundary conditions are continuity of displacement η , its radial derivative η_r , momentum M and shear force V at the boundaries [see Eq. (A.27) and (A.28) in Appendix]. Note that spatial part of the incident planar wave can be written as $\eta^{inc} = a_0 \exp(ik_0 x) = a_0 \sum_{n=-\infty}^{\infty} i^n J_n(k_0 r) \exp(in\theta)$, where a_0 is the amplitude of the wave, $k_0^4 = \rho_0 h \omega^2 / D_0$ and D_0 is the constant flexural rigidity outside the cloak.

In order to quantitatively analyze the efficiency of the cloak, we calculate the scattering cross section which corresponds to the energy scattered to the infinity. The scattered displacement field is $\eta^{sc} = \eta - \eta^{inc}$, where η^{inc} is the incident plane wave [see Fig. 3]. The scattered far field amplitude $f(\theta)$ is defined through [see e.g. 22]

$$\eta^{sc} = \frac{a_0}{\sqrt{2r}} e^{i(k_0 r - \pi/4)} f(\theta) + \mathcal{O}(1/\sqrt{r})$$
(8)

and the total scattering cross section is $\sigma^{sc} = 1/2\pi \oint |f(\theta)|^2 d\theta$.

IV. RESULTS

We present here a side-by-side comparison of the surface elevation η , scattered displacement field η^{sc} and scattered far field amplitude $f(\theta)$ for three cases: i. in the absence of cloak, ii. with the claimed linear cloak of [19, 20], and iii. with our nonlinear cloak. We implement the linear cloak according to Eq. (4) of [19] for a cloak size of b/a = 4. We approximate the cloak N = 15 concentric layers that are homogeneous but anisotropic and then we use two isotropic and homogeneous sub-layers to approximate each of the 15 layer [see e.g. 19, 20, 27]. The resulted layers of isotropic and homogeneous materials approximates the anisotropic inhomogeneous cloak. We do the same for the nonlinear cloak but in this case according to equation 5. The result for a linear cloak design [19, 20] and the nonlinear cloak for the range of the frequencies of f = 200Hz-500Hz alongside with the case when there is no cloak is shown in Fig. 2 and 3 for comparison.

Looking at the displacement field in Fig. 2, both of the cloaks seems to be effective at lower frequencies. In fact, the linear cloak *looks* to be more effective than the nonlinear cloak in f=200 Hz. More specifically, the nonlinear cloak has less scattering downstream of the cylinder compared to the linear cloak and the linear cloak have less scattering upstream of the cylinder compared to the nonlinear case [see Fig. 3]. By increasing the frequency, we observe that downstream scattering of the nonlinear cloak is much better in preserving the wave shape; while at the upstream of the cylinder the linear cloak has less scattering. We would like to mention that although linear cloak looks to be more efficient in lower frequencies, the proposed nonlinear cloak has much less scattering in a broader range of frequencies. The snapshot of scattered displacement field η^{sc} corresponding to





FIG. 2. Nondimensionalized displacement field η/a_0 for a nonlinear cloak (left column), without any cloak (middle column), and linear cloak (right column). Each row corresponds to a different frequency: 200Hz (first row), 300Hz (second row), 400Hz (third row) and 500Hz (the last row). Coordinates are nondimensionalized with radius of the inner cylinder a and the cloak size for both linear and nonlinear cloak is b/a = 4. The cloaks are approximated with N = 15 layers of homogeneous anisotropic materials with each layer composed of two sub-layers made up of different homogeneous and isotropic materials. For a direct comparison with Stenger et al. [20], Frequencies are obtained using the values of $h = 1mm, a = 1.5cm, \rho = 2000 kg/m^3$ and $D_0 = 0.1037 Nm^2$. A nonlinear cloak shows a consistent cloaking efficiency for different frequencies, while the performance of linear cloak drops significantly as the frequency increases. The scattered field is also shown in Fig. 3. For a quantitative comparison of performance, see Fig. 5 and 6.

Fig. 2 is shown in Fig. 3. This figure explicitly shows that the linear cloak compared to other cases scatters far more energy in the downstream of cylinder for all ranges of frequency. However, the nonlinear cloak scatters less energy in the downstream, and the upstream scattering increases partly as the frequency increases. We would like to point out that the downstream scattering is re-

FIG. 3. Nondimensionalized scattered field $\eta^{sc}/a_0 = (\eta - \eta^{inc})/a_0$ where η^{inc} is the incident wave, for a nonlinear cloak (left column), without any cloak (middle column), and linear cloak (right column). Each row corresponds to a different frequencies: 200Hz (first row), 300Hz (second row), 400Hz (third row) and 500Hz (the last row). Coordinates are nondimensionalized with radius of the inner cylinder *a* and the cloak size for both linear and nonlinear cloak is b/a = 4.

lated to the change in wave rays' path followed by waves in the cloak: the path is longer than that of rays traveling straightforwardly in an isotropic thin plate [c.f. 28]. This path length difference results in phase difference between the plane waves propagating in an isotropic plate and the wave field of a cloaked system in the downstream of the cloak. Note that the change in rays' path length in a nonlinear cloak is smaller than a linear cloak which causes less scattering in the downstream of the cloak.

In order to demonstrate more quantitatively how superposition of the scattered field and the incident wave results in graphs of Fig. 2, we show in Fig. 4 the nondimensionalized scattered amplitude η^{sc}/a_0 along the center-line of the simulation domain, i.e. $\theta = 0, \pi$, at the frequency of f = 300Hz for three cases : i. with no cloak (---), ii. with a linear cloak (---), and iii. with



FIG. 4. Nondimensionalized scattered amplitude η^{sc}/a_0 along the *x*-axis at the frequency f = 300Hz for (i) without a cloak (---), (ii) with a linear cloak (---), and (iii) with our nonlinear cloak (---). We also show the nondimensionalized incident wave amplitude η^{inc} (·--).

our nonlinear cloak (- - -). We also show, for the sake of comparison, the incident wave field $(\cdot \cdot \cdot)$. In forming the final picture of waves over the water surface, the relative phases of incident and scattered waves play significant roles. For example, downstream of the cylinder (r/a > 1) and in the absence of a clock, scattered waves and incident wave have an almost π -radian phase difference, and hence nearly cancel each other resulting in a clearly distinguishable shadow zone in Fig. 2, middle column. A perfect cloak must have zero scattered waves outside the cloak, and for a cloak to be considered effective, it must have a scattering field with an amplitude that is at least *smaller* than that without any cloak. Clearly, our nonlinear cloak satisfies the latter condition, whereas the scattering field of a linear cloak does not (amplitude of scattering field on the downstream is clearly larger than the amplitude of scattering field in the absence of the cloak).

In order to quantitatively test the effectiveness of the cloaks, we present in Fig. 5 the absolute value of the scattered amplitude $|f(\theta)|$ at different angles for both of the cloaks at different frequencies f = 200Hz-500Hz. As is seen, in all of the frequencies, although in upstream of the cylinder the linear cloak is reducing the amount of energy scattered to infinity compared to the nonlinear case, far more energy is scattered in the downstream of the cylinder in the linear cloak, even larger than the case when there is no cloak. Therefore, in all cases the linear cloak scatters *more* energy to the downstream compared to when the cloak does not exist. Our nonlinear cloak consistently achieves a lower scattering in all angles with no exception.

In order to quantitatively see the net effect of the cloak in terms of the total energy scattered to the infinity, we also calculate and plot the total scattering cross section σ^{sc} (Fig. 6). In the frequencies bellow 200Hz, although the linear cloak scatters more energy at certain angles (at the downstream side of the cylinder) compared to the case with no cloak [see Fig. 5], the total energy scattered to the infinity is smaller than the case with no cloak. At frequencies above 200Hz the linear cloak both scatters far more energy at the back of the cylinder and also in total.



FIG. 5. Polar plot of absolute value of scattered amplitude $|f(\theta)|$ for the linear cloak (- - -), without cloak (----), and with nonlinear cloak (- - -) for different frequencies of (a) 200Hz, (b) 300Hz, (c) 400Hz and (d) 500Hz.



FIG. 6. Nondimensionalized total scattering cross section with wavelength i.e. $k\sigma^{sc}$ for the a range of frequencies for 3 different cases of without cloak, with a linear cloak and with a nonlinear cloak.

Note that, the total scattering cross section for the nonlinear cloak stays always smaller than the case without any cloak. This underlines the broadband effectiveness of our proposed nonlinear cloak.

V. CONCLUSION

In summary, we presented here the design of a perfect broadband cloak for flexural waves. Since the governing equations for flexural waves are not form-invariant, the traditional cloak design methodology through linear transformation optics scheme does not apply here. We therefore employed a nonlinear transformation and matched, term-by-term, the transformed equation with the true governing equation for an inhomogeneous and orthotropic plate equation. We showed rigorously that the resulting cloak is perfect. We also presented an adoption of our perfect cloak obtained under more restrictive physical constraints that make the design more amenable for experimental investigations. These constraints are that cloak can only include a finite number of concentric layers of homogeneous materials and that only modulus of elasticity can be variable from layer to layer. We presented this approximate cloak, and showed via direct simulation that this experimentally realizable cloak of such type has a consistent performance in all spatial directions, and also has a broad bandwidth of high efficiency.

The nonlinear cloak proposed in here is combination of layers of homogeneous and isotropic materials, which are amenable to physical fabrication and testing and real-life application [e.g. potentially in cloaking against earthquakes 29, 30]. The nonlinear transformation proposed here, may be applied for other types of the waves to soften the required material properties. For instance, in electromagnetism, with this nonlinear cloak, we can remove one degree of the freedom and keep permeability (permittivity) as a constant in cloaking for transverse magnetic (or electric) waves.

APPENDIX

1. Governing Equations

Assuming an orthotropic and inhomogeneous plate, under pure bending and in the absence of in-plane forces, we have [31]

$$\frac{\partial^2 M_R}{\partial R^2} + \frac{2}{R} \frac{\partial M_R}{\partial R} + \frac{2}{R} \frac{\partial^2 M_{\Theta R}}{\partial R \partial \Theta} + \frac{2}{R^2} \frac{\partial M_{\Theta R}}{\partial \Theta} + \frac{1}{R^2} \frac{\partial^2 M_{\Theta}}{\partial \Theta} + \frac{1}{R} \frac{\partial^2 M_{\Theta}}{\partial R} - \frac{1}{R} \frac{\partial M_{\Theta}}{\partial R} + \rho_0 h \omega^2 \eta = 0, \quad (A.9)$$

where M_R, M_{Θ} and $M_{R\Theta}$ are the bending moments, ρ_0 is the density of the plate, h is the thickness and ω is the frequency of the wave. Bending moments M_R, M_{Θ} and $M_{R\Theta}$ are found as

$$M_R = -D_R \left[\frac{\partial^2 \eta}{\partial R^2} + \nu_\Theta \left(\frac{1}{R} \frac{\partial \eta}{\partial R} + \frac{1}{R^2} \frac{\partial^2 \eta}{\partial \Theta^2} \right) \right],$$
(A.10a)

$$M_{\Theta} = -D_{\Theta} \left(\frac{1}{R} \frac{\partial \eta}{\partial R} + \frac{1}{R^2} \frac{\partial^2 \eta}{\partial \Theta^2} + \nu_R \frac{\partial^2 \eta}{\partial R^2} \right), \quad (A.10b)$$

$$M_{R\Theta} = -2D_K \frac{\partial}{\partial R} \left(\frac{1}{R} \frac{\partial \eta}{\partial \Theta} \right), \qquad (A.10c)$$

where $\eta(R, \Theta)$ is the out-of-plane displacement, D_R, D_{Θ} are the flexural rigidities in the R, Θ directions respectively, D_K is the shearing rigidity and ν_R, ν_{Θ} are the Poisson ratios in the radial and tangential directions. Note that the radial and tangential rigidities D_R, D_{Θ} satisfy the symmetry relation $D_R\nu_{\Theta} = D_{\Theta}\nu_R$. Using these relations and defining $D_{R\Theta} = 2D_K + D_R\nu_{\Theta}$, equation (A.9) simplifies to [18]

$$D_{R}\frac{\partial^{4}\eta}{\partial R^{4}} + \frac{D_{\Theta}}{R^{4}}\frac{\partial^{4}\eta}{\partial\Theta^{4}} + \frac{2D_{R\Theta}}{R^{2}}\frac{\partial^{4}\eta}{\partial R^{2}\partial\Theta^{2}} + \left(\frac{2D_{R}}{R} + 2\frac{\partial D_{R}}{\partial R}\right)\frac{\partial^{3}\eta}{\partial R^{3}} + \left(\frac{2}{R^{2}}\frac{\partial D_{R\Theta}}{\partial R} - \frac{2D_{R\Theta}}{R^{3}}\right)\frac{\partial^{3}\eta}{\partial R\partial\Theta^{2}} + \left[\frac{1}{R^{2}}\frac{\partial}{\partial R}\left(R^{2}\frac{\partial D_{R}}{\partial R}\right) + \frac{1}{R}\frac{\partial(D_{R}\nu_{\Theta})}{\partial R} - \frac{D_{\Theta}}{R^{2}}\right]\frac{\partial^{2}\eta}{\partial R^{2}} + \left(\frac{\partial^{2}\eta}{\partial\Theta^{2}}\right)\left(\frac{2D_{R\Theta}}{R^{4}} - \frac{2}{R^{3}}\frac{\partial D_{R\Theta}}{\partial R} + \frac{2D_{\Theta}}{R^{4}} - \frac{1}{R^{3}}\frac{\partial D_{\Theta}}{\partial R} + \frac{1}{R^{2}}\frac{\partial^{2}(D_{R}\nu_{\Theta})}{\partial R^{2}}\right) + \left(\frac{D_{\Theta}}{R^{3}} - \frac{1}{R^{2}}\frac{\partial D_{\Theta}}{\partial R} + \frac{1}{R}\frac{\partial^{2}(D_{R}\nu_{\Theta})}{\partial R^{2}}\right)\frac{\partial\eta}{\partial R} - \rho_{0}h\omega^{2}\eta = 0.$$
(A.11)

Assuming a constant rigidity $D_R = D_{\Theta} = D_{R\Theta} = D_0$, equation (A.11) simplifies to the famous biharmonic plate's equation as

$$D_0 \Delta^2 \eta - \rho_0 h \omega^2 \eta = 0. \tag{A.12}$$

Now as we aim to cloak a circular region A_c with radius a, with a cloak of outer radius b co-centered with A_c , we use the following transformation \mathcal{F} to map the area $0 \leq R \leq b$ to the cloaking region $a \leq r \leq b$ [11]

$$\mathcal{F}: \left\{ \begin{array}{l} r = \sqrt{(1 - a^2/b^2) R^2 + a^2}, \ 0 \le R \le b, \\ \theta = \Theta. \end{array} \right.$$
 (A.13)

The Jacobi of the transformation ${\mathcal F}$ in the polar coordinate is

$$\mathbf{F} = \sqrt{1 - a^2/b^2} \begin{pmatrix} \sqrt{r^2 - a^2}/r & 0\\ 0 & r/\sqrt{r^2 - a^2} \end{pmatrix}_{\substack{(\mathbf{e}_r, \mathbf{e}_\theta) \\ (A.14)}},$$

where $\{\mathbf{e}_r = \mathbf{E}_R, \mathbf{e}_{\theta} = \mathbf{E}_{\Theta}\}$. Note that det $\mathbf{F} = \sqrt{1 - a^2/b^2} \neq 1$.

Transforming the governing equation (A.12), we obtain [see Lemma 2.1 in Ref. 25]

$$\tilde{\nabla}^2 \tilde{\nabla}^2 \eta - \rho_0 h \omega^2 \eta = 0, \qquad (A.15)$$

where

$$\tilde{\nabla}^2 = \left(1 - \frac{a^2}{b^2}\right) \left[\frac{1}{r}\frac{\partial}{\partial r}\left(\frac{r^2 - a^2}{r}\frac{\partial}{\partial r}\right) + \frac{1}{r^2 - a^2}\frac{\partial^2}{\partial\theta^2}\right]$$
(A.16)

Defining the following parameters

$$D'_r = \alpha^2 \mathcal{A}^2(r) D_0, \qquad (A.17a)$$

$$D'_{\theta} = \alpha^2 (1/\mathcal{A}(r))^2 \ D_0,$$
 (A.17b)

$$D'_{r\theta} = \alpha^2 D_0, \tag{A.17c}$$

$$\nu_{\theta}' = \frac{1}{\alpha^2 \mathcal{A}^2(r)} \left[\mathcal{B}(r) - 4 \log \mathcal{A}(r) \right].$$
 (A.17d)

where $\alpha = 1 - a^2/b^2$ and $\mathcal{A}(r) = 1 - a^2/r^2$ and $\mathcal{B}(r) = 3(r/a) \log\left(\frac{r-a}{r+a}\right) - 2a^2/(r^2 - a^2)$. We can further expand and simplify equation (A.15) as

$$\begin{split} D_r' \frac{\partial^4 \eta}{\partial r^4} &+ \frac{D_{\theta}'}{r^4} \frac{\partial^4 \eta}{\partial \theta^4} + \frac{2D_{r\theta}}{r^2} \frac{\partial^4 \eta}{\partial r^2 \partial \theta^2} + \\ \left(\frac{2D_r'}{r} + 2\frac{\partial D_r'}{\partial r}\right) \frac{\partial^3 \eta}{\partial r^3} &+ \left(\frac{2}{r^2} \frac{\partial D_{r\theta}'}{\partial r} - \frac{2D_{r\theta}'}{r^3}\right) \frac{\partial^3 \eta}{\partial r \partial \theta^2} + \\ \left[\frac{1}{r^2} \frac{\partial}{\partial r} \left(r^2 \frac{\partial D_r'}{\partial r}\right) + \frac{1}{r} \frac{\partial (D_r \nu_{\theta}')}{\partial r} - \frac{D_{\theta}'}{r^2} + \mathcal{C}(r)\right] \frac{\partial^2 \eta}{\partial r^2} + \frac{\partial^2 \eta}{\partial \theta^2} \\ \left(\frac{2D_{r\theta}'}{r^4} - \frac{2}{r^3} \frac{\partial D_{r\theta}'}{\partial r} + \frac{2D_{\theta}'}{r^4} - \frac{1}{r^3} \frac{\partial D_{\theta}'}{\partial r} + \frac{1}{r^2} \frac{\partial^2 (D_r' \nu_{\theta}')}{\partial r^2}\right) + \\ \left(\frac{D_{\theta}'}{r^3} - \frac{1}{r^2} \frac{\partial D_{\theta}'}{\partial r} + \frac{1}{r} \frac{\partial^2 (D_r' \nu_{\theta}')}{\partial r^2} + \mathcal{D}(r)\right) \frac{\partial \eta}{\partial r} - \rho_0 h \omega^2 \eta = 0 \\ (A.18) \end{split}$$

where

$$\mathcal{C}(r) = \frac{1}{2a^2} \left(\frac{a}{r}\right)^8 \mathcal{L}(r) D_0, \qquad (A.19a)$$
$$\mathcal{D}(r) = \frac{3}{a^3} \left(\frac{a}{r}\right)^{11} \frac{5 - 12(r/a)^2 + 8(r/a)^4}{(1 - a^2/r^2)^2} D_0, \quad (A.19b)$$

where

$$\mathcal{L}(r) = \frac{6 - 10(r/a)^2 - 2(r/a)^4 + 3(r/a)^6}{1 - (a/r)^2} + \frac{3}{2} \left(\frac{r}{a}\right)^7 \log\left(\frac{r-a}{r+a}\right).$$
(A.20)

The transformed equation i.e. equation (A.18), matches with the inhomogeneous and orhotropic plate's equation i.e. equation (A.11), with the rigidities and the Poisson ratio as defined in (A.17). The only difference is in the second order term $\partial^2 \eta / \partial r^2$ and the first order term $\partial \eta / \partial r$ with the coefficients defined in (A.19). Note that these remaining terms goes to zero as long as $r \gg a$, i.e. the penetration depth of the wave into the cloak is small.

These extra terms in equation (A.18) can also be interpreted as an additional pre-stress force \mathbf{N} and body force \mathbf{S} as

$$N_{rr} = \mathcal{C}(r), \qquad N_{\theta\theta} = N_{r\theta} = 0, \qquad (A.21)$$

$$S_r = -\mathcal{D}(r), \qquad S_\theta = 0. \tag{A.22}$$

Note that the pre-stress force N and the body force S should satisfy the following constraint as

$$\nabla \mathbf{N} + \mathbf{S} = 0. \tag{A.23}$$

2. Numerical Solution

Our cloak is composed of concentric layers of homogeneous materials with a clamped boundary condition at the inner most layer. We have plane incident waves and we aim to find the cloak's response to these incoming waves.

For each layer, we expand the solution in that layer using spectral methods as

$$\eta^{(i)}(r,\theta) = \operatorname{Re}\left\{e^{\mathrm{i}\omega t}\sum_{n=-\infty}^{\infty} Z_n(k_i r)e^{\mathrm{i}n\theta}\right\},\qquad(A.24)$$
$$Z_n(k_i r) = A^{(i)}J_n(k_i r) + B^{(i)}J_n(k_i r)$$

$$+ C_n^{(i)} Y_n(k_i r) + E_n^{(i)} K_n(k_i r)$$
(A.25)

where $k_i^4 = \rho h \omega^2 / D_i$ with D_i being the flexural rigidity of the layer. Here, $J_n(.), Y_n(.)$ and $I_n(.), K_n(.)$ are respectively Bessel and modified Bessel functions of the first and second kind and $A_n^{(i)}, B_n^{(i)}, C_n^{(i)}, E_n^{(i)}$ are constants that are later found satisfying the boundary conditions.

Outside of the cloak, since the solution should remain finite and satisfy the radiation condition, the solution can be written as

$$\eta^{\text{out}}(r,\theta) = \operatorname{Re}\left\{e^{\mathrm{i}\omega t}\sum_{n=-\infty}^{\infty} \mathcal{Z}_n(k_0 r)e^{\mathrm{i}n\theta}\right\},\$$
$$\mathcal{Z}_n(k_0 r) = F_n H_n^{(1)}(k_0 r) + G_n K_n(k_0 r) + a_0 \mathrm{i}^n J_n(k_0 r),$$
(A.26)

where $H_n^{(1)}(.)$ is the Hankel function of the first kind, $k_0^4 = \rho h \omega^2 / D_0$ and D_0 being the flexural rigidity outside of the cloak. Note that $a_0 i^n J_n(k_0 r)$ represents the plane incident wave. Boundary conditions at the boundary of each layer is the continuity of η , its radial derivative η_r and also the continuity of momentum and shear force as

$$M_r = -D\left[\frac{\partial^2 \eta}{\partial r^2} + \nu \left(\frac{1}{r}\frac{\partial \eta}{\partial r} + \frac{1}{r^2}\frac{\partial^2 \eta}{\partial \theta^2}\right)\right], \qquad (A.27)$$

$$V = Q_r + \frac{1}{r} \frac{\partial M_{r\theta}}{\partial \theta},\tag{A.28}$$

where

$$Q_r = -D\frac{\partial}{\partial r}\nabla^2\eta, \qquad (A.29)$$

$$M_{r\theta} = -D(1-\nu)\frac{\partial}{\partial r}\left(\frac{1}{r}\frac{\partial\eta}{\partial\theta}\right).$$
 (A.30)

Using the above boundary conditions at each layer and also the clamped boundary condition $\eta = \eta_r = 0$ at the boundary of the inner most layer, we can solve for the unknowns.

3. Discussion on transformation

In this section we show that the offset ϵ introduced earlier, is equivalent to transforming the region $\varepsilon \leq R \leq b$ to the cloaking region $a \leq r \leq b$, where ε is a small nonzero number. A transformations with a constant Jacobian, mapping the area $\varepsilon \leq R \leq b$ to the cloaking area $a \leq r \leq b$ can be written as

$$\mathcal{F}: \begin{cases} r = \sqrt{c_1 R^2 + c_2}, & \varepsilon \le R \le b, \\ \theta = \Theta, \end{cases}$$
(A.31)

where

$$c_1 = \frac{b^2 - a^2}{b^2 - \varepsilon^2}, \qquad c_2 = a^2 - \varepsilon^2 \frac{b^2 - a^2}{b^2 - \varepsilon^2}.$$
 (A.32)

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Note that when $\varepsilon = 0$, equation (A.31) reduces to equation (A.13). The offset we used in Fig. 1 is equivalent to

picking the value of ε as $\varepsilon/a = 0.175$.

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