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Phys. Rev. A **97**, 063820 — Published 11 June 2018

DOI: [10.1103/PhysRevA.97.063820](https://doi.org/10.1103/PhysRevA.97.063820)

# Dissipative vs dispersive coupling in quantum opto-mechanics: squeezing ability and stability

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(Dated: May 24, 2018)

## Abstract

Generation of squeezed light and optomechanical instability for dissipative type of opto-mechanical coupling is theoretically addressed for a cavity with the input mirror, serving as a mechanical oscillator, or an equivalent system. The problem is treated analytically for the case of resonance excitation or small detunings, mainly focusing on the bad cavity limit. A qualitative difference between the dissipative and purely dispersive coupling is reported. In particular, it is shown that, for the purely dissipative coupling in the bad cavity regime, the backaction is strongly reduced and the squeezing ability of the system is strongly suppressed, in contrast to the case of purely dispersive coupling. It is also shown that, for small detunings, stability diagrams for the cases of the purely dispersive and dissipative couplings are qualitatively identical to within the change of the sign of detuning. The results obtained are compared with those from the recent theoretical publications.

PACS numbers: 42.50.Lc, 42.50.Wk, 07.10.Cm, 42.50.Ct

## I. INTRODUCTION

Cavity quantum optomechanics is a rapidly developing branch of quantum optics which allows for exploration of fundamental issues of quantum mechanics and paves the way for numerous applications, e.g. in high-precision metrology and gravitational-wave defection<sup>1</sup>. The work horse of cavity optomechanics is the so-called *dispersive coupling* originating from the dependence of the cavity resonance frequency on the position of a mechanical oscillator. However as pointed out by Elste et al<sup>2</sup>, the dispersive coupling does not provide the complete description of the optomechanical interaction. To fill the gap, those authors have introduced the so-called *dissipative coupling*, which can be interpreted in terms of the dependence of the cavity damping rate on the mirror position. Since then, manifestations of this coupling have been addressed both theoretically<sup>3-9</sup> and experimentally<sup>10-12</sup>.

Studying and harnessing this effect experimentally seems to be a tough task since it is difficult to find a situation where the dissipative coupling can be distinguished from the stronger dispersive coupling. Such situation was theoretically identified by Xuereb et al<sup>3</sup> and later experimentally explored by Sawadsky et al<sup>12</sup> in their setup based on a modified Michelson-Sagnac interferometer where the relative strength of the dispersive and dissipative coupling can be tuned so that the latter can be not dominated by the former. One should note that though the dissipative coupling is a higher order effect compacted to that dispersive the absolute value of the dissipative coupling constant of such device is not so small. Using the results from Ref. 12, one finds that, for this device, the dispersive coupling constant on average (far from some special points) obeys the standard estimate for an optomechanical cavity<sup>1</sup> while the dissipative coupling constant is smaller than the former by a factor, which is about the amplitude transmission coefficient of the effective mirror, characterizing the decay rate of the interferometer.

On the theoretical side, a number of interesting consequences of this coupling have been revealed, e.g. a remarkable Fano effect due to the interference between the dispersive and dissipative couplings<sup>2,3</sup>. For the case of dissipative coupling, detailed zero-temperature numerical simulations of the optomechanical instability as well as of the squeezing and photon correlation spectra have been done by Kilda and Nunnenkamp<sup>7</sup>. One of their results is an interesting possibility of simultaneous squeezing and sideband cooling in the resolved-sideband regime.

As mentioned in the literature<sup>1,2</sup>, the bad cavity regime (cavity decay rate greater than the mechanical frequency) is of special interest since, in this regime, the dissipative coupling may provide the ground state cooling of a mechanical oscillator while that dispersive may not<sup>2</sup>. A focussed finite-temperature analytical treatment of some manifestations of the dissipative coupling in this regime was recently published by Qu and Agraval<sup>6</sup>. These authors argued that the manifestations of the dissipative coupling they addressed (squeezing spectra and stability conditions) can be quantitatively recovered from the corresponding results for dispersive coupling by replacing the coupling constant of the dispersive coupling with that of the dissipative coupling. However, some equations and results from Ref. 6 are in a conflict with other publications. Specifically, the stability criterion offered in that paper is incompatible with the results of simulations by Kilda and Nunnenkamp<sup>7</sup>. There is also discrepancy in the Langevin equations and input-output relations between Ref. 6 and Refs. 2,3.

In view of the above, it seems reasonable to revisit analytically the squeezing generation and the stability conditions of a system with dissipative optomechanical coupling. This is the subject of the present paper, where we focus on the bad cavity limit. We have addressed the problem in terms of the Hamiltonian originally introduced by Elste et al<sup>2</sup>, which is the one exploited in all theoretical papers on the topic. The Hamiltonian describes a one-sided cavity with the input mirror serving as a mechanical oscillator, as well as equivalent systems, including the Michelson-Sagnac interferometer-based setup<sup>3,12</sup>.

Our analysis demonstrates that, in the bad cavity regime, the purely dissipative coupling manifests itself qualitatively different from the dispersive one. Namely, in this regime, the backaction due to the dissipative coupling is strongly reduced, vanishing in the limit  $\Delta\omega/\gamma \rightarrow 0$  where  $\Delta\omega$  and  $\gamma$  are the frequency band of interest around the laser frequency and the cavity decay rate, respectively. Specifically, in this regime, the dissipative coupling constant is effectively multiplied by a factor of the order  $\Delta\omega/\gamma$ . This effect reveals an additional weakness of the optomechanical interaction in this regime.

Our analytical calculations also provide an explanation for the qualitative difference between the optomechanical stability diagrams for the purely dispersive and purely dissipative coupling cases<sup>7</sup>. None of our results supports those published by Qu and Agraval<sup>6</sup>.

The paper is organized as follows. First, we address, in the simplest terms, the aforementioned reduction of the oscillator-cavity coupling for dissipative case in the bad cavity

limit. Then, we address the implications of this reduction for the squeezing ability of the optomechanical cavity, controlled solely by the dissipative coupling. We finish the paper with the stability analysis of the system. Throughout the presentation we illustrate how the behavior of the system controlled by purely dissipative coupling can be mapped onto the system controlled by purely dispersive coupling and then utilize the well known results for the latter system.

## II. REDUCTION OF BACKACTION FORCE IN ONE-SIDED CAVITY

We start from the Hamiltonian originally introduced by Elste et al<sup>2</sup>, which describes a one-sided cavity with the input mirror, serving as a mechanical oscillator, and equivalent systems

$$\mathbf{H} = \hbar\omega_c \mathbf{a}^\dagger \mathbf{a} + \hbar\omega_m \mathbf{b}^\dagger \mathbf{b} - \hbar g_\omega \mathbf{a}^\dagger \mathbf{a} (\mathbf{b}^\dagger + \mathbf{b}) - i\hbar \sqrt{\frac{\gamma}{2\pi\rho}} \sum_q (\mathbf{a}^\dagger \mathbf{c}_q - \mathbf{c}_q^\dagger \mathbf{a}) [1 - g_\gamma (\mathbf{b}^\dagger + \mathbf{b})/\gamma], \quad (1)$$

where  $\hbar$  is the Planck constant,  $\omega_c$ ,  $\gamma$ , and  $\mathbf{a}$  are the resonance frequency, decay rate, and the ladder Bose operator of the cavity field, respectively, while  $\omega_m$  and  $\mathbf{b}$  are the resonance frequency and the ladder Bose operator for the mechanical oscillator.  $\mathbf{c}_q$  are ladder operators for the electromagnetic bath (the bath Hamiltonian is omitted),  $\rho$  is its density of states in the frequency range of interest [per unit frequency]. Here  $g_\omega$  and  $g_\gamma$  are the coupling constants for the dispersive and dissipative interactions, respectively. The mechanical oscillator is assumed to be coupled to a thermal bath (the Hamiltonian containing the degrees of freedom of the thermal bath is also omitted).

The cavity is driven with a laser light of the frequency  $\omega_L$ . Assuming the Markovian bath, one derives in a standard way the Langevin equations describing both the dispersive and dissipative couplings, which (in the frame rotating with the laser frequency) read

$$\frac{\partial \mathbf{a}}{\partial t} + \{\gamma/2 + i[\omega_c - \omega_L]\} \mathbf{a} = \sqrt{\gamma} \mathbf{A}_{\text{in}} + (\mathbf{b}^\dagger + \mathbf{b}) [g_\gamma (\mathbf{a} - \mathbf{A}_{\text{in}}/\sqrt{\gamma}) + ig_\omega \mathbf{a}] \quad (2)$$

$$\frac{\partial \mathbf{b}}{\partial t} + \left(\frac{\gamma_m}{2} + i\omega_m\right) \mathbf{b} = \sqrt{\gamma_m} \mathbf{b}_{\text{in}} - \frac{g_\gamma}{\sqrt{\gamma}} (\mathbf{a}^\dagger \mathbf{A}_{\text{in}} - \mathbf{A}_{\text{in}}^\dagger \mathbf{a}) + ig_\omega \mathbf{a}^\dagger \mathbf{a}. \quad (3)$$

where  $\mathbf{A}_{\text{in}}$  is the operator of the input electromagnetic field and  $\mathbf{b}_{\text{in}}$  is the operator of the thermal mechanical noise. We assume the driving laser field to be strong so that the operators  $\mathbf{a}$ ,  $\mathbf{b}$ , and  $\mathbf{A}_{\text{in}}$  consist of constant [in the rotating reference frame] classical parts,  $a_0$ ,  $b_0$ ,  $A_0$ , and operator parts,  $\delta \mathbf{a}$ ,  $\delta \mathbf{b}$ , and  $\delta \mathbf{A}_{\text{in}}$ , describing fluctuations.

We are particularly interested in the fluctuations for the case of purely dissipative coupling, i.e. at  $g_\omega = 0$  and  $g_\gamma \neq 0$ . The equations describing the fluctuations are routinely obtained by linearization of Eqs. (2) and (3). In the notations where  $\delta$ 's are dropped from  $\delta \mathbf{a}$ ,  $\delta \mathbf{b}$ , and  $\delta \mathbf{A}_{\text{in}}$  for simplicity, these equations read<sup>2</sup>:

$$\frac{\partial \mathbf{a}}{\partial t} + \{\gamma/2 - i\Delta\} \mathbf{a} = \sqrt{\gamma} \mathbf{A}_{\text{in}} + g_\gamma (a_0 - A_0/\sqrt{\gamma})(\mathbf{b}^+ + \mathbf{b}) \quad (4)$$

$$\frac{\partial \mathbf{b}}{\partial t} + \left( \frac{\gamma_m}{2} + i\omega_m \right) \mathbf{b} = \sqrt{\gamma_m} \mathbf{b}_{\text{in}} + \mathbf{F}_{\text{diss}} \quad (5)$$

where  $\Delta = \omega_L - \omega_c$  and the operator of the backaction force (to within a factor of some physical dimension) has a form (c.f. Eq.(5) from Ref. 2):

$$\mathbf{F}_{\text{diss}} = -\frac{g_\gamma}{\sqrt{\gamma}} (a_0^* \mathbf{A}_{\text{in}} + A_0 \mathbf{a}^+ - \mathbf{A}_{\text{in}}^+ a_0 - A_0^* \mathbf{a}). \quad (6)$$

Here  $\mathbf{A}_{\text{in}}$  describes the vacuum noise

$$[\mathbf{A}_{\text{in}}(t), \mathbf{A}_{\text{in}}^+(t')] = \delta(t - t') \quad [\mathbf{A}_{\text{in}}(t), \mathbf{A}_{\text{in}}(t')] = 0 \quad \langle \mathbf{A}_{\text{in}}^+(t) \mathbf{A}_{\text{in}}^+(t') \rangle = 0, \quad (7)$$

$\langle \dots \rangle$  and  $[\dots, \dots]$  denoting the ensemble averaging and the commutator, respectively.

The set of equations (4)-(6) should be appended with the steady state equation for  $a_0$ :

$$\{\gamma/2 - i\Delta\} a_0 = \sqrt{\gamma} A_0. \quad (8)$$

Now we would like to demonstrate, in the simplest situation, an important result of this paper. Namely, that the backaction due to the dissipative coupling is strongly suppressed in the bad cavity regime. For simplicity, we consider the lowest approximation in optomechanical coupling constants, though, as will be shown in Subsect.III B, the same conclusion holds for an arbitrary strength of optomechanical couplings.

Let us evaluate the backaction force in the case where the detuning and the time derivative (in Eqs. (4)) are neglected, the requirement, equivalent to the bad cavity limit. By setting  $a_0$  real and taking into account that  $A_0 \approx \sqrt{\gamma} a_0/2$  we find

$$\mathbf{F}_{\text{diss}} \approx -\frac{g_\gamma}{\sqrt{\gamma}} a_0 [\mathbf{A}_{\text{in}} - \mathbf{A}_{\text{in}}^+ - (\mathbf{a} - \mathbf{a}^+) \sqrt{\gamma}/2]. \quad (9)$$

Now taking into account that, to within our approximations, Eq.(4) yields

$$\mathbf{A}_{\text{in}} - \mathbf{A}_{\text{in}}^+ \approx (\mathbf{a} - \mathbf{a}^+) \sqrt{\gamma}/2, \quad (10)$$

we see that Eq.(9) implies a strong reduction of the backaction force in the bad cavity limit. This is in a sharp contrast with the backaction force due to the dispersive coupling, which, in the lowest order in the coupling constant, according to (3) reads:

$$\mathbf{F}_{\text{disp}} = ig_{\omega}a_0(\mathbf{a} + \mathbf{a}^+) \approx i\frac{2g_{\omega}a_0}{\sqrt{\gamma}}(\mathbf{A}_{\text{in}} + \mathbf{A}_{\text{in}}^+). \quad (11)$$

One can readily trace the origin of this difference. In the case of dissipative coupling, the vacuum electromagnetic noise reaches the mechanical oscillator via two channels: from the cavity field and directly (see Eq.(9)). In the bad cavity regime, a distractive interference between those channels takes place, resulting in the backaction force reduction. At the same time, in the case of dispersive coupling, nothing similar can happen because now the vacuum electromagnetic noise reaches the mechanical oscillator via the only channel - the cavity field (see Eq.(11)). Evidently, the above effect entails a reduction of squeezing ability of the dissipative coupling in the bad cavity limit since it is the backaction that provides mixing of the quadratures of the output light, leading to squeezing of the optimized quadrature. We will address this issue in detail in Subsect.III B below.

### III. SQUEEZING

In this section, we compare the squeezing abilities of the dispersive and dissipative couplings for a single-ended cavity with a mechanical oscillator as the input mirror (or an equivalent system) in the bad cavity limit. To make presentation more transparent we consider the case of a resonance excitation of the cavity. In Subsect.III A we reproduce the well known result for the dispersive coupling as a benchmark. In Subsect.III B we map the squeezing problem for the case of the dissipative coupling onto that for the dispersive one and, without further calculations, we recover the results for the dissipative coupling from those obtained in Subsect.III A.

#### A. Dispersive coupling

To obtain the squeezing spectrum of the generalized quadrature of the light backscattered from the cavity, we perform the Fourier transforms on all operators describing fluctuations

and obtain for the frequency components, e.g.

$$\mathbf{a}(\omega) = \frac{1}{\sqrt{2\pi}} \int dt e^{i\omega t} \mathbf{a}(t) \quad (12)$$

$$\mathbf{a}^+(\omega) = \frac{1}{\sqrt{2\pi}} \int dt e^{-i\omega t} \mathbf{a}^+(t). \quad (13)$$

We introduce the following quadrature operators

$$\mathbf{Q}(\omega) = [\mathbf{b}(\omega) + \mathbf{b}^+(-\omega)]/2 \quad \mathbf{P}(\omega) = -i[\mathbf{b}(\omega) - \mathbf{b}^+(-\omega)]/2$$

$$\mathbf{X}(\omega) = [\mathbf{a}(\omega) + \mathbf{a}^+(-\omega)]/2 \quad \mathbf{Y}(\omega) = -i[\mathbf{a}(\omega) - \mathbf{a}^+(-\omega)]/2$$

$$\mathbf{X}_{\text{in}}(\omega) = \mathbf{A}_{\text{in}}(\omega) + \mathbf{A}_{\text{in}}^+(-\omega) \quad \mathbf{Y}_{\text{in}}(\omega) = -i[\mathbf{A}_{\text{in}}(\omega) - \mathbf{A}_{\text{in}}^+(-\omega)],$$

where the Fourier components of the noise operators satisfy the relationships:

$$\langle \mathbf{X}_{\text{in}}(\omega) \mathbf{X}_{\text{in}}(\omega') \rangle = \langle \mathbf{Y}_{\text{in}}(\omega) \mathbf{Y}_{\text{in}}(\omega') \rangle = i \langle \mathbf{Y}_{\text{in}}(\omega) \mathbf{X}_{\text{in}}(\omega') \rangle = \quad (14)$$

$$-i \langle \mathbf{X}_{\text{in}}(\omega) \mathbf{Y}_{\text{in}}(\omega') \rangle = \delta(\omega + \omega').$$

Note that, following Ref. 13, for convenience of calculations, we use non-identical definitions of the quadratures for the field in the cavity and for those outside it.

Starting from the standard linearized Langevin equations for fluctuations in the optomechanical cavity controlled by the dispersive coupling, e.g. from Ref.<sup>4</sup>, we rewrite those in the quadrature variables (rotating reference frame), dropping the frequency argument  $\omega$  wherever it is not confusing:

$$(\gamma/2 - i\omega)\mathbf{X} = \frac{\sqrt{\gamma}}{2}\mathbf{X}_{\text{in}} \quad (15)$$

$$(\gamma/2 - i\omega)\mathbf{Y} = \frac{\sqrt{\gamma}}{2}\mathbf{Y}_{\text{in}} + G_\omega \mathbf{Q} \quad (16)$$

$$\chi(\omega)^{-1}\mathbf{Q} = \sqrt{\gamma_{\text{m}}}\mathbf{W} + G_\omega \mathbf{X} \quad (17)$$

$$\chi(\omega)^{-1} = \frac{1}{\omega_{\text{m}}} [\omega_{\text{m}}^2 - \omega^2 - i\gamma_{\text{m}}\omega] \quad (18)$$

$$G_\omega = 2g_\omega a_0 \quad (a_0 \text{ is set real}). \quad (19)$$

Restricting ourselves to the high-temperature case  $T \gg \hbar\omega_{\text{m}}$ , in Eq.(17), we introduce the mechanical noise in the simplest Markovian form via the operator  $\mathbf{W}(\omega)$ , which satisfies the relationship:<sup>14</sup>

$$\langle \mathbf{W}(\omega) \mathbf{W}(\omega') \rangle \approx (n_{\text{th}} + 1/2)\delta(\omega + \omega') \quad n_{\text{th}} \approx T/\omega_{\text{m}}, \quad (20)$$



$T$  being the temperature in the energy units. Here we have also neglected the small renormalization of the mechanical frequency when passing from  $\mathbf{b}$  to  $\mathbf{Q}$  operators.

The squeezing of the backscattered light in the dissipative coupling regime is evaluated by the variance of the generalized quadrature

$$\mathbf{Z}(\omega, \theta) = \mathbf{X}_{\text{out}}(\omega) \cos \theta + \mathbf{Y}_{\text{out}}(\omega) \sin \theta \quad (21)$$

where  $\mathbf{X}_{\text{out}}$  and  $\mathbf{Y}_{\text{out}}$  are defined by the standard input-output relations

$$\mathbf{X}_{\text{in}} + \mathbf{X}_{\text{out}} = 2\sqrt{\gamma}\mathbf{X} \quad (22)$$

$$\mathbf{Y}_{\text{in}} + \mathbf{Y}_{\text{out}} = 2\sqrt{\gamma}\mathbf{Y}. \quad (23)$$

Taking the bad cavity limit  $\gamma \gg \omega_m$  and keeping in mind that, in the situation of interest,  $|\omega|$  and  $\omega_m$  are of the same order, the above formulae readily yield the explicit input-output relations

$$\mathbf{X}_{\text{out}} = \mathbf{X}_{\text{in}} \quad (24)$$

$$\mathbf{Y}_{\text{out}} = \mathbf{Y}_{\text{in}} + \frac{4G_\omega}{\sqrt{\gamma}}\chi(\omega) \left[ \sqrt{\gamma_m}\mathbf{W} + \frac{G_\omega}{\sqrt{\gamma}}\mathbf{X}_{\text{in}} \right]. \quad (25)$$

Then, straightforward calculations yield

$$\langle \mathbf{Z}(\omega, \theta) \mathbf{Z}(\omega', \theta) \rangle = \delta(\omega + \omega') S_{ZZ}(\omega, \theta) \quad (26)$$

$$S_{ZZ}(\omega, \theta) = 1 + M \sin^2 \theta + N \sin \theta \cos \theta \quad (27)$$

$$M = 16n_{\text{ba}}\gamma_m^2 |\chi(\omega)|^2 (n_{\text{th}} + n_{\text{ba}} + 1/2) \quad (28)$$

$$N = 8n_{\text{ba}}\gamma_m \text{Re}[\chi(\omega)]. \quad (29)$$

Here

$$n_{\text{ba}} = \frac{G_\omega^2}{\gamma_m \gamma} \quad (30)$$

is the optomechanical cooperativity or, alternatively, the noise added by the backaction to the intrinsic noise of the mechanical oscillator (normalized to the number of mechanical quanta).

Since it is the even part of the spectral power density that is used for characterization of squeezing, hereafter, for this variable we will keep only the frequency-even parts.

The result of minimization of the spectral power density of the generalised quadrature  $S_{ZZ}(\omega, \theta)$  with respect to the "mixing" angle  $\theta$ ,  $S_m(\omega)$ , reads

$$S_m(\omega) = 1 - \frac{N^2/2}{\sqrt{M^2 + N^2} + M}. \quad (31)$$

The results of further minimization (with respect to  $\omega$ ) can be presented in a transparent form if we focus on the frequency range close to the mechanical frequency. Specifically for  $\delta = \omega_m - \omega$  satisfying the following inequalities

$$|\delta| \ll \omega_m \quad (32)$$

and

$$\frac{|\delta|/\gamma_m}{n_{th} + n_{ba} + 1/2} \ll 1. \quad (33)$$

These inequalities allow us to neglect the  $N^2$ -term under the square root in Eq.(31) and present this equation in the form

$$S_m(\omega) = \frac{n_{th} + 1/2}{n_{ba} + n_{th} + 1/2} + \frac{n_{ba}}{n_{ba} + n_{th} + 1/2} \frac{\text{Im}[\chi(\omega)]^2}{|\chi(\omega)|^2} \quad (34)$$

Keeping in mind that in the frequency range of interest

$$\frac{\text{Im}[\chi(\omega)]^2}{|\chi(\omega)|^2} \approx \frac{1}{1 + (2\delta/\gamma_m)^2},$$

Eq. (34) can be further simplified to get:

$$S_m = \frac{n_{th} + 1/2}{n_{ba} + n_{th} + 1/2} + \frac{n_{ba}}{n_{ba} + n_{th} + 1/2} \frac{1}{1 + (2\delta/\gamma_m)^2}. \quad (35)$$

This equation implies that, in the potentially quite wide frequency range defined by the conditions

$$1 \ll |\delta|/\gamma_m \ll n_{th} + n_{ba}, \omega_m/\gamma_m,$$

the squeezing parameter of the optimized generalized quadrature approaches a limiting value of

$$S_0 = \frac{n_{th} + 1/2}{n_{ba} + n_{th} + 1/2}. \quad (36)$$

This result is consistent with the well known result by Fabre et al<sup>15</sup>.

## B. Dissipative coupling

Now we keep all settings used in the previous Subsection the same, except we consider the dissipative coupling instead of the dispersive one. The linearized Langevin equations for this system in terms of the ladder operators can be found in Refs. 2,4. We rewrite these equations in the quadrature variables:

$$(\gamma/2 - i\omega)\mathbf{X} = \frac{\sqrt{\gamma}}{2}\mathbf{X}_{\text{in}} + G_{\gamma}\mathbf{Q} \quad (37)$$

$$(\gamma/2 - i\omega)\mathbf{Y} = \frac{\sqrt{\gamma}}{2}\mathbf{Y}_{\text{in}} \quad (38)$$

$$\chi(\omega)^{-1}\mathbf{Q} = \sqrt{\gamma_{\text{m}}}\mathbf{W} + G_{\gamma}(\mathbf{Y} - \mathbf{Y}_{\text{in}}/\sqrt{\gamma}) \quad (39)$$

where

$$G_{\gamma} = g_{\gamma}a_0. \quad (40)$$

The difference between the structure of the above mechanical equation and that for the dispersive coupling, Eq.(17), is clearly seen: in the mechanical equation corresponding to dissipative coupling, we observe a direct contribution of the vacuum noise, which is responsible for the strong suppression of the backaction in the bad cavity limit, discussed in Sect. II. The input-output relations<sup>3</sup> also differ from those for the dispersive coupling by an explicit appearance of the mechanical variable:

$$\mathbf{X}_{\text{in}} + \mathbf{X}_{\text{out}} = 2\sqrt{\gamma}\mathbf{X} - \frac{4}{\sqrt{\gamma}}G_{\gamma}\mathbf{Q} \quad \mathbf{Y}_{\text{in}} + \mathbf{Y}_{\text{out}} = 2\sqrt{\gamma}\mathbf{Y}. \quad (41)$$

Based on the above equations, we find the following explicit input-output relations, keeping only the lowest non-vanishing terms in  $\omega/\gamma$ :

$$\mathbf{Y}_{\text{out}} = \mathbf{Y}_{\text{in}} \quad (42)$$

$$\mathbf{X}_{\text{out}} = \mathbf{X}_{\text{in}} + \frac{4G_{\gamma}\beta(\omega)}{\sqrt{\gamma}}\chi(\omega) \left[ \sqrt{\gamma_{\text{m}}}\mathbf{W} + \frac{G_{\gamma}\beta(\omega)}{\sqrt{\gamma}}\mathbf{Y}_{\text{in}} \right] \quad \beta(\omega) = \frac{2i\omega}{\gamma}. \quad (43)$$

Relation (25) is worth commenting on. The small factor  $\omega/\gamma$  enters this relation two times: inside and outside the brackets. Its first appearance describes the backaction reduction discussed in terms of Langevin equations in Sect.II. The second appearance describes the reduced ability of the system to read the position of the oscillator. This is a result of cancelation due to the presence of the  $\mathbf{Q}$  operator in the input-output relations (41). Thus,

for measurements, we cannot profit from the reduction of the backaction since the informative signal is also reduced. Such symmetry is well-expected since if the signal were not reduced it would mean a possibility of backaction-free measurements, which, for the system considered, are forbidden by the Heisenberg uncertainty principle.

Comparing this set of equations with the set (25) and (24), we note that those sets are equivalent to each other to within swapping

$$\mathbf{X} \Leftrightarrow \mathbf{Y} \quad G_\omega \Leftrightarrow G_\gamma \beta(\omega).$$

This correspondence implies that the well known results reproduced in the previous Subsection, Eq.(34), can be, with a proper modification, applied to the system with the dissipative coupling. Specifically, we can write

$$S_m = \frac{n_{\text{th}} + 1/2}{n_{\text{ba1}} + n_{\text{th}} + 1/2} + \frac{n_{\text{ba1}}}{n_{\text{ba1}} + n_{\text{th}} + 1/2} \frac{\text{Im}[\chi(\omega)]^2}{|\chi(\omega)|^2}. \quad (44)$$

where

$$n_{\text{ba1}} = \frac{G_\gamma^2}{\gamma_m \gamma} \left( \frac{2\omega}{\gamma} \right)^2, \quad (45)$$

plays a role of the optomechanical cooperativity.

There exists one more difference between the dispersive and dissipative system: swapping  $\mathbf{X} \Leftrightarrow \mathbf{Y}$  leads to the permutation  $\cos \theta \Leftrightarrow \sin \theta$  in Eq.(27). That means that, after the substitution  $G_\omega \Leftrightarrow G_\gamma \beta(\omega)$ , the expressions for the maximal squeezing are identical, but the expressions for the optimal angle are not.

Similarly to the previous case, we thus find that, in the frequency range defined by conditions

$$1 \ll \delta/\gamma_m \ll n_{\text{th}} + n_{\text{ba1}}, \omega_m/\gamma_m \quad \delta = |\omega - \omega_m|,$$

the squeezing parameter of the optimized generalized quadrature approaches a limiting value of

$$S_0 = \frac{n_{\text{th}} + 1/2}{n_{\text{ba1}} + n_{\text{th}} + 1/2}. \quad (46)$$

The result for the optomechanical cooperativity of the system, (45), in combination with (44) suggests that the bad cavity regime is unfavorable for the squeezing ability of the system.

Our results are in a conflict with those obtained by Qu and Agrawal<sup>6</sup> for squeezing in the dissipative system at resonance in the bad cavity limit. In the notations of our paper, the

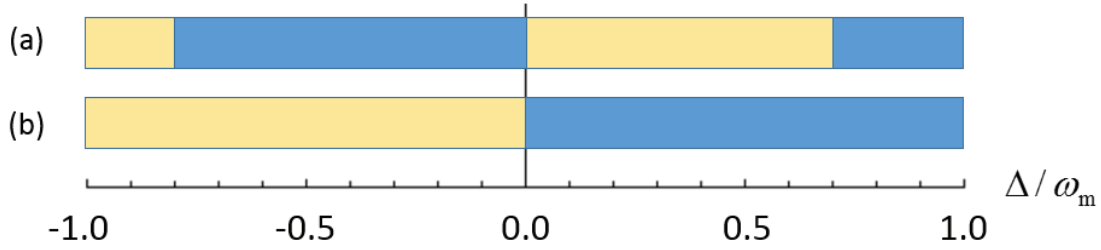


FIG. 1. Stability diagram for a one-sided cavity with the input mirror, serving as mechanical oscillator, or an equivalent system, according to Kilda and Nunnenkamp<sup>7</sup>. Dark areas show instability intervals in terms of the normalized detuning  $\Delta/\omega_m$  ( $\Delta = \omega_L - \omega_c$  - detuning,  $\omega_m$  - mechanical frequency). (a)- purely dissipative coupling, (b) - purely dispersive coupling. In notations of our paper, the model settings are  $G_\omega = 1.2\gamma$ ,  $G_\gamma = -0.3\gamma$ ,  $\gamma_m/\omega_m = 10^{-5}$ ,  $\gamma/\omega_m = 0.3$ ;  $\gamma$  - decay rate of the cavity,  $\gamma_m$  - mechanical decay rate.

result by Qu and Agraval misses the important factor of  $\beta(\omega)$  in the definition of  $n_{ba1}$ . It does not seem feasible to fully clarify the origin of this disparity. However, one problem is clearly seen in Ref. 6: the vital term with the  $\mathbf{Q}$  operator in the input-output relations (41) is neglected.

#### IV. STABILITY

For the case of purely dispersive coupling, the stability of a one-sided cavity with the input mirror serving as the mechanical oscillator has been treated by many authors (see e.g. Ref. 15). At the same time, for the case of purely dissipative coupling, described by Hamiltonian offered by Elste et al<sup>2</sup>, the stability problem was addressed only recently: numerically by Nunnenkamp with coworkers<sup>4,5,7</sup> and analytically by Qu and Agraval<sup>6</sup>, in the bad cavity limit. According to the numerical calculations<sup>7</sup> (see Fig.1a), the system with purely dissipative coupling is unstable with respect to small red detuning, in contrast to the small-blue-detuning instability for the dispersive coupling (see Fig.1b). In contrast, according to analytical calculations by Qu and Agraval<sup>6</sup>, a small blue detuning can destabilize both systems, while a small red detuning is safe.

To resolve this disparity, we show how, in the limit of small detunings, the instability problems for those two couplings can be mapped onto each other.

For the dispersive system the instability problem reduces to the analysis of the linear set of equations following, e.g. from Ref. 7:

$$\begin{aligned}
\dot{\mathbf{X}} &= -\frac{\gamma}{2}\mathbf{X} - \Delta\mathbf{Y} \\
\dot{\mathbf{Y}} &= \Delta\mathbf{X} - \frac{\gamma}{2}\mathbf{Y} + G_\omega\mathbf{Q} \\
\dot{\mathbf{Q}} &= \omega_m\mathbf{P} \\
\dot{\mathbf{P}} &= G_\omega\mathbf{X} - \omega_m\mathbf{Q} - \gamma_m\mathbf{P}.
\end{aligned} \tag{47}$$

where the dot means the time derivative. (In this section, the bold-type letters are used for the time-dependent operators, not their Fourier transforms as in the previous sections.) The application of Routh-Hurwitz criterion<sup>16</sup> for small detuning (only linear terms in  $\Delta$  are kept) readily yields the stability condition:

$$\frac{\Delta}{\omega_m} < \left(\frac{\gamma}{G_\omega}\right)^2 \frac{\gamma}{\omega_m} Q \left[ 1 + 4Q \left(\frac{\omega_m}{\gamma}\right)^3 + 16 \left(\frac{\omega_m}{\gamma}\right)^4 \right] \quad Q = \gamma_m/\omega_m, \tag{48}$$

here we have also neglected  $\gamma_m$  compared to other frequency-units parameters. This relation is consistent with the well known results from Ref. 15. It also complies well with the results of modelling by Kilda and Nummenkamp<sup>7</sup>, yielding the instability threshold  $\frac{\Delta}{\omega_m} = +4*10^{-3}$ , c.f. Fig.1b. On the practical side, this condition means that the system is formally stable at the resonant excitation, however, a small blue detunings jeopardize its stability.

For the dissipative coupling, using the results from Ref. 7, we find, in the linear approximation in detuning  $\Delta$ <sup>17</sup>, a similar set of equations for the stability analysis:

$$\begin{aligned}
\dot{\mathbf{X}} &= -\frac{\gamma}{2}\mathbf{X} - \Delta\mathbf{Y} + G_\gamma\mathbf{Q} \\
\dot{\mathbf{Y}} &= \Delta\mathbf{X} - \frac{\gamma}{2}\mathbf{Y} \\
\dot{\mathbf{Q}} &= \omega_m\mathbf{P} \\
\dot{\mathbf{P}} &= G_\gamma\mathbf{Y} - \omega_m\mathbf{Q} - \gamma_m\mathbf{P}.
\end{aligned} \tag{49}$$

Comparing the instability problem given by Eq.(47) with that given by Eq.(49) we clearly see that those two are equivalent to each other to within swapping

$$G_\omega \Leftrightarrow G_\gamma \quad \Delta \Leftrightarrow -\Delta.$$

The transformation  $\Delta \Leftrightarrow -\Delta$  explains the complementarity of the instability regions seen in Fig.1a and b. Specifically, it implies that, in contrast to the dispersive-coupling system, in the dissipative-coupling system, small red detunings lead to instability, in contrast to the result obtained by Qu and Agrawal<sup>6</sup>.

## V. DISCUSSION AND CONCLUSIONS

We have theoretically addressed the squeezing generation and optomechanical instability due to dissipative coupling for a one-sided optomechanical cavity or an equivalent system. The Hamiltonian introduced by Elste et al<sup>2</sup> has been used as the starting point of the analysis. We have focused on the case of the resonance or small (compared to the decay rate) detuning and the bad cavity regime. We have identified a remarkable qualitative difference in the manifestations of the purely dissipative and purely dispersive couplings. We have shown that, in this regime, for the purely dissipative coupling, the backaction is strongly reduced while the squeezing ability of the system is strongly suppressed, in contrast to the case of purely dispersive coupling. This implies that, for the dissipative coupling system, this regime is extremely unfavorable for the squeezing purposes. Our results also provide qualitative explanation for the numerical results on the optomechanical instability obtained by Nunnenkamp and coworkers<sup>4,5,7</sup>. Specifically, we have demonstrated that, for small detuning, the stability diagrams for the cases of purely dispersive and purely dissipative coupling are complementary.

Our results for the case of purely dissipative coupling apply to the setup based on a modified Michelson-Sagnac interferometer<sup>3</sup>, where the relative strength of the dispersive and dissipative coupling can be tuned so that the purely dissipative-coupling regime becomes experimentally feasible<sup>12</sup>.

## ACKNOWLEDGMENTS

A.K.T. acknowledges fruitful discussions with Sergey Fedorov. This work was supported by the Army Research Office with funding from the DARPA, Innovation Fund Denmark through the EU program EUREKA, John Templeton Foundation, and the Russian Founda-

tion for Basic Research under project 18-02-00648.

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- <sup>17</sup> The linear approximation in  $\Delta$  can be readily made in the Langeven equation for the field ladder operator where, at finite detuning, the parameter  $G_\gamma$  acquires a complex factor. The absolute value of this factor differs from 1 by a corection of the order of  $\Delta/\gamma$ . Taking into account this corection would result in the apearence of higher powers of  $\Delta$  in the final stability condition, so that it is neglected, leading to the set of equations for the quadatures, Eqs. (49).